

National Technical University of Athens  
Naval and Marine Technology Master Program  
Offshore Structures Section  
School of Naval Architecture and Marine Engineering



# Hydrodynamic Analysis of a Porous Cylindrical Body

Master Thesis

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Supervisor  
Professor: Ioannis K. Chatjigeorgiou

Athens, 2022

# Acknowledgments

First and foremost, I would like to express my gratitude to my supervisor, Professor Ioannis K. Chatjigeorgiou who gave me the opportunity for collaboration in such an interesting project. The remarks and the advice he gave me were always very helpful and enlightening.

I would also like to thank Assistant Professor Dimitris Konispoliatis for always being willing to provide help and knowledge. His guidance is what made this project possible.

Last but not least, I would like to thank my family and my friends for every kind of support they provide.

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### **Abstract**

In this study, the problems of diffraction and radiation of water waves from a surface porous cylindrical body are considered. The body is free floating in a constant water depth, and is consisted of a porous sidewall and impermeable bottom. Within the realms of linear potential theory, a three dimensional solution based on the method of matched eigenfunction expansions is applied in order to predict the velocity potential of the flow field surround the body. Numerical results are presented and discussed concerning the effect the porosity, the draught and the radius have on the cylindrical body's hydrodynamic forces and moments. It is revealed by the results that the porosity and the dimensions of the body play a significant role in either reducing or increasing the exciting wave loads.

Εθνικό Μετσόβιο Πολυτεχνείο

Δ.Π.Μ.Σ Ναυτική και Θαλάσσια Τεχνολογία

Κατεύθυνση: Θαλάσσιες Κατασκευές και Εχμετάλλευση Υδρογονανθράκων

Σχολή Ναυπηγών Μηχανολόγων Μηχανικών

## Υδροδυναμική Ανάλυση Διάτρητου Κυλινδρικού Σώματος

Μεταπτυχιακή Εργασία

**Μπουφίδη Ιφιγένεια**

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Αθήνα, 2022

### Περίληψη

Στην παρούσα μελέτη εξετάζονται τα προβλήματα της διάθλασης και ακτινοβολίας κυματισμών, από την παρουσία πορώδους κυλινδρικού σώματος. Το σώμα πλέει ελεύθερο σε σταθερό βάθος, διαπερνά την ελεύθερη επιφάνεια και αποτελείται από διάτρητη εξωτερική επιφάνεια και αδιαπερατό πυθμένα. Στα πλαίσια της γραμμικής θεωρίας κυματισμών, εφαρμόστηκε τρισδιάστατη μέθοδος ανάπτυξης ιδιοσυναρτήσεων ώστε να υπολογιστεί το δυναμικό ταχύτητας του πεδίου ροής που περιβάλλει το σώμα. Στην συνέχεια, παρουσιάζονται και σχολιάζονται αριθμητικά αποτελέσματα που αφορούν την επίδραση του πορώδους, της ακτίνας και του βυθίσματος του διάτρητου κυλινδρικού σώματος στα υδροδυναμικά φορτία. Γίνεται εμφανές, ότι τόσο το πορώδες όσο και οι διαστάσεις του σώματος παίζουν σημαντικό ρόλο στην μείωση ή αύξησή των φορτίων.

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# 1 Introduction

## 1.1 Perforated Structures in Offshore Engineering

Perforated structures are commonly used in a variety of applications in offshore engineering. One of the most common uses is the construction of perforated breakwaters and seawalls. The way in which the porosity of these breakwaters dissipates wave energy and reduces wave loads has been thoroughly studied in the literature, as discussed in section (1.2), making them a very suitable choice for protecting coasts and infrastructure. Some typical examples of perforated breakwaters and seawalls are shown in the following figures:



Figure 1.1: Wolnae fishing port of Busan [1]



Figure 1.2: Chilam fishing port of Busan [1]



Figure 1.3: A bottom-sitting, surface-piercing pile breakwater along the coast of Singapore [2]

In Figures [1.1]-[1.2] the type of breakwater (or seawall) depicted is the one first studied by Jarlan [3] and now it bears his name. In fact, they are partially or fully perforated-wall caisson breakwaters. In their simplest form, they consist of a perforated front wall, in the cases depicted Figures [1.1]-[1.2] circular perforated caisson wall, and a vertical impermeable back-wall.

Whereas, in Figure [1.3] the perforated breakwater has no impermeable back-wall. This type of breakwater is particular useful in applications where the wave circulation is of essence. This could be the case, in near-shore applications, e.g harbours or marinas, where it is important to keep the water of being stagnant. Furthermore, the natural wave circulation that is allowed in this type of structures can help improve the water quality and maintain a healthy ecosystem. Additional advantages of this type of perforated breakwater, are the sediment transport along them and the low construction costs when the water is relatively deep [4].

Another important application of perforated/porous structures in offshore engineering is the use of fish cages. The high environmental impact of aquaculture in near-shore applications, and as the availability of coastal areas becomes progressively more limited, the need for offshore aquaculture has become more urgent. In the following Figures [1.4],[1.5] some examples of offshore aquaculture with the use of fish cages are shown:



Figure 1.4: Fish cages in Velfjorden, Brønnøy, Norway.



Figure 1.5: The Samkna offshore fish farm site off the Qatar coast in the Persian Gulf.



## 1.2 Literature Review

The subject of modelling wave interaction with porous structures have been widely studied in the literature through different approaches. In [5]- [6] the Computational Fluid Dynamics (CFD) approach is followed. In this approach, the flow through the porous surface is modelled explicitly. This requires a numerical grid which solves both the small-scale characteristics of the openings in the porous surface and the large scales required for the wave propagation, resulting in a high number of grid cells and correspondingly high computational times. As an alternative, and in order to solve flow fields complex structures, numerical approaches such as Boundary Element Methods (BEM) are required. A few examples of BEM models for wave interaction with porous bodies are [7]- [8]. For less complex structures such as cylinders, plates and vertical barriers (e.g breakwaters), the potential flow methods based on linear wave theory are often preferred. In such approaches, the flow through the porous surface openings is not explicitly modelled, but is rather parameterised on the grounds of the pressure drop as a function of the flow velocity, with a linear or quadratic pressure-velocity relationship.

As aforementioned, the most common use of porous structures on offshore engineering is the use of porous breakwaters. [9]- [10] are just a few, that investigated the wave reflection, transmission and energy dissipation inside such structures, with the application of eigenfunction expansions in the flow field in front and inside the porous medium, as well as behind the breakwater. Using the same method, [11] proved that the wave loads on a semi-porous cylindrical breakwater were significantly reduced due to its porosity. Several subsequent studies followed, which theoretically and experimentally examined the induced wave flow in a porous breakwater and its ability to scatter wave energy and minimize the environmental impact [12]- [13]. In addition, porous plates or meshes are sometimes used to absorb waves in narrow channels [14]- [15] and have also been studied in the literature because of their ability to freely exchange water or marine species. More specifically, in [16]- [17] in order to analyse small amplitude surface waves generated by a porous plate oscillations, a porous wavemaker theory was developed. While, in [18] the reflection and transmission of small amplitude waves produced by a rigid porous plate that is bottom-mounted in a channel of constant depth, is investigated. The solution of scattering and radiation problems, in both submerged and floating elastic porous plates, was further expanded by the work of [19]- [20], as well as by, [21] and [22] who studied the hydroelastic behaviour of flexible submerged porous plates, in terms of wave energy absorption.

The interaction of waves with porous cylindrical bodies is another issue that has been particularly studied in the literature. The wave interaction of a porous free surface piercing cylindrical body with a coaxial inner impermeable cylinder was initially studied in [23], concluding in the fact that the existing of porous outer surface leads to reduced hydrodynamic forces in the inner cylinder. In order to evaluate wave kinematics around a protected cylindrical impermeable pile, an analytical methodology was developed and presented in [24]. Furthermore, the wave interaction with an array of bottom-mounted surface-piercing porous cylinders was investigated in [25] using an eigenfunction expansion method. This was further extended in [26] where the body interacting with the waves, is a free floating cylinder and its proven that its porosity directly affects the hydrodynamic loads. In [27] the problem of wave diffraction around a cylinder with porous upper wall and an inner column was studied, and concluded in that an increased porous coefficient lead to a reduced wave elevation around the body. The case of wave diffraction around a group of dual porous cylinders was numerically and experimentally studied in [28]. Similarly, numerical and experimental results regarding the hydrodynamics of a con-

centric twin perforated circular cylinder system were presented in [29]. Bao et al [30], applied the Haskind relations in order to evaluate the the eigenfunctions values of the fluid domain inside a semi-submerged porous circular cylinder and in order. This concluded to the derivation of a new damping term created by porosity. The case of an array of freely floating, porous, circular cylinders was investigated in [31]. In [32] the wave interaction with a single truncated cylinder was investigated and a theoretical method, that was also validated experimentally was developed. Theoretical and experimental results regarding an array of porous circular cylinders with impermeable bottom, supplemented by horizontal, porous plates inside the bodies were presented in [33]. In addition, an array of partially porous, surface-piercing cylindrical bodies were examined in [34] with the development of a 3D numerical model. An optimal ratio of porous to impermeable portion of surface was adopted in order to minimize the hydrodynamic impact. In [35] they study the impact of geometrical characteristics and porous coefficient on the hydrodynamic loads on a truncated cylinder, and the proper selection of said values that will lead in reduced hydrodynamic loads. Furthermore, in [36], [37], the effect of linear and quadratic resistance laws on porous bodies of arbitrary shape was examined. Cong et al [38] applied the image method to study the effect of a vertical breakwater on the hydrodynamics of a bottom-mounted, porous cylinder that pierces the free surface and is placed in front of the wall. It was concluded that the presence of the porous cylinder can reduce the amplified wave effects that derive from the presence of the wall. In [39] the hydrodynamics of a moored porous cylindrical body are studied.

Another widely investigated subject in the literature of porous cylinders is the water wave interaction with fish-farm cages. In the recent decade some notable studies are [40]- [41], where a porous flexible circular cylinder is simulating the fish net. In order to further investigate wave interaction with fish cages, several computational models have been developed. For example, in [42] in the framework of potential theory and without taking under consideration the existence of currents, a slender-body theory is presented to simulate the wave effects on a floating fish cage. This study was further extended in [43] with the development of a numerical model capable of calculating viscous waves and current loads on fish nets. Continually, this was further developed in [44], [45] to taking into account the induced wave and current mooring loads on the fish cage nets. The mooring characteristics and behaviour of a elastic fish cage with a cubic shape was validated with experimental results and presented in [46]. In [47] the hydrodynamic response of a porous flexible circular-cylinder fish cage in regular waves was analytically studied where in [48] a semi analytical a model was developed describing the wave field around a submerged cylindrical fish net cage. Furthermore, in [] a numerical model to describe wave–current interactions with a moored flexible cylindrical cage was developed. It was further validated with numerical finite element model results. In [49] a semi-analytical solution to describe the interactions between waves and a flexible cylindrical net cage, the deformation of which is governed by the membrane vibration equation of cylindrical shells, is presented.

### 1.3 Thesis Objective

In the present study, a free floating, free surface piercing, porous cylindrical body is examined. The porous outer surface is assumed to be inflexible while the bottom of the body to be impermeable. The linearised problems of diffraction and radiation around the porous body are solved in the frequency domain according to the theoretical approach described thoroughly in the following chapters. The hydrodynamic problems of surge, heave and roll are resolved with

the implementation of matched axisymmetric eigenfunction expansions method. This method allows the subdivision of the flow field around and inside the body into coaxial, ring shaped fluid regions categorized by the numerals *I, II, III* (see 2.1), in each of which different series representations of the velocity potential are made. These series expansions are selected in such a way, that the kinematic boundary condition at the horizontal walls of the body, the kinematic condition on the sea bed, the radiation one at infinity and the linearised condition at the free surface, are a priori satisfied. The various solutions, that derived with the use of Galerkin method, are matched by the requirements for continuity of the hydrodynamic pressure and radial velocity along the vertical boundaries of adjacent fluid regions, as well as by fulfilling appropriate boundary conditions of the porous vertical walls of the body, that derived from the application of Darcy's linear law. The accuracy of the present study is validated by comparisons of data available in the existing literature. Lastly, the effect of porosity and radius on the body's exciting forces and moments, hydrodynamic coefficients of added mass and damping, and motions is investigated.

## 1.4 Thesis Structure

Chapter 1 briefly reviews the use and types of porous structures in offshore engineering, summarizes and reviews relevant to the present thesis literature, and presents the aim and structure of the thesis.

Chapter 2 presents the formulation of the problem, describes the boundary value problem, presents the analytical solutions for the hydrodynamic coefficients of added mass and damping and the solutions for the excitation forces and moments and describes the equation of motion of the body.

Chapter 3 presents results regarding the effect of porous coefficient  $G$ , body's radius and draught, on the permeable body's exciting forces and moment and hydrodynamic coefficients of added mass and damping, as well as on the body's motions. Furthermore, the numerical method used in this study is validated by comparisons of data, available in the existing literature.

Chapter 4 presents the conclusions drawn from the present study and suggests areas for future research.

Lastly, bibliographical references are listed.

## 2 Problem Formulation

### 2.1 Boundary Values Problem

In the present study the examined body is assumed to be, a free floating cylinder that pierces the free surface, has an impermeable thin bottom and a permeable outer surface. It is subjected to the action of regular waves of circular frequency  $\omega$  and linear amplitude  $(H/2)$  which propagate at a zero angle in a constant water depth of  $d$ . The radius of the body is denoted by  $a$ , the distance of the bottom of the body to the seabed is denoted by  $h$ , while the distance of the bottom porous surface (the top of the impermeable bottom) is denoted by  $h_1$ . The problem is described by a cylindrical coordinate system  $(r, \theta, z)$  originating at the seabed and coinciding with the axis of symmetry of the body (see Figure 1). Furthermore, the fluid is assumed to be inviscid and incompressible while the flow irrotational.

It is assumed that the water domain is divided into three regions as follows, see Figure [2.1]:

- **flow region I**  
 $r \geq a; 0 \leq z \leq d;$
- **flow region II**  
 $0 \leq r \leq a; h_1 \leq z \leq d;$
- **flow region III**  
 $0 \leq r \leq a; 0 \leq z \leq h;$

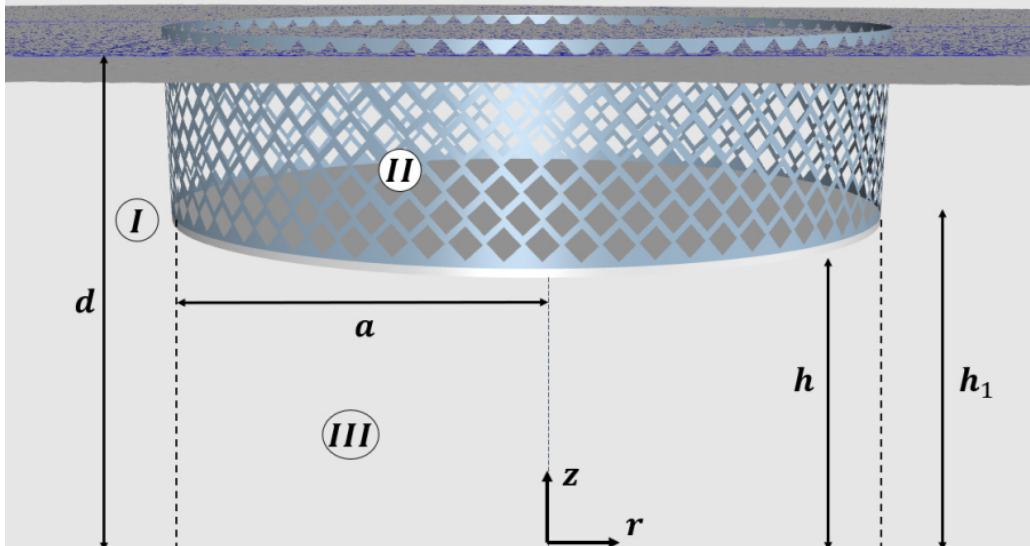


Figure 2.1: A 3D representation of the examined permeable vertical cylindrical body [39]

The assumption of irrotational flow allows the flow in each fluid region to be described in terms

of velocity potential as follows:

$$\Phi^k(r, \theta, z; t) = \text{Re}[\phi^k(r, \theta, z)e^{-i\omega t}], \quad k = I, II, III \quad (2.1)$$

The linearisation of the problem leads to the decomposition of the potential function into three terms: the velocity potential due to undisturbed incident harmonic wave  $\phi_0$ , the velocity potential related to the scattered disturbance of the incident waves from the body  $\phi_s$ , the velocity potential of waves radiated due to the motion of the body  $\phi_7$ . Further, we denote the velocity potential due to the diffracted waves  $\phi_D$ . In the cases of  $\phi_s$  and  $\phi_D$  the body is considered to be stationary at its mean static equilibrium, whereas in the case of radiation potential  $\phi_7$  the body is considered to be performing forced oscillations at its motions of freedom in still liquid. The following equations describe the aforementioned velocity potentials and the relations between them.

The velocity potential of every fluid region can be described as:

$$\phi^k(r, \theta, z) = \phi_0(r, \theta, z) + \phi_7^k(r, \theta, z) + \sum_{j=1,3,5} \dot{\xi}_{j0} \phi_j^k(r, \theta, z), \quad k = I, II, III \quad (2.2)$$

where  $\dot{\xi}_{j0}$  is the complex velocity amplitude of body motion in the  $j$ -th direction.

While the diffraction problem is described by:

$$\phi_D^k(r, \theta, z) = \phi_0(r, \theta, z) + \phi_7^k(r, \theta, z), \quad k = I, II, III \quad (2.3)$$

The velocity potential of the undisturbed incident wave can be expressed using Jacobis's expansion as:

$$\phi_0(r, \theta, z) = -i\omega \frac{H}{2} \frac{\cosh(kz)}{k \sinh(kd)} \sum_{m=0}^{\infty} \epsilon_m i^m J_m(kr) \cos(m\theta) \quad (2.4)$$

where where  $J_m$  denotes the  $m$ -th order Bessel function of first kind and  $m$  the Neumann's symbol:

$$\epsilon_m = \begin{cases} 1, & m = 0 \\ 2, & m \geq 1 \end{cases} \quad (2.5)$$

The circular frequency  $\omega$  and wave number  $k$  are related by the dispersion equation:

$$\omega^2 = gk \tanh(kd) \quad (2.6)$$

In accordance to Equation (2.4) the diffraction potential of the flow field around the body can be expressed as:

$$\phi_D^k(r, \theta, z) = -i\omega \frac{H}{2} \sum_{m=0}^{\infty} \epsilon_m i^m \Psi_{Dm}^k(r, z) \cos(m\theta) \quad (2.7)$$

Similar to Equation (2.7) the radiation potentials can be described as:

$$\phi_1^k(r, \theta, z) = \Psi_{11}^k(r, z) \cos(\theta) \quad (2.8)$$

$$\phi_3^k(r, \theta, z) = \Psi_{30}^k(r, z) \quad (2.9)$$

$$\phi_5^k(r, \theta, z) = \Psi_{51}^k(r, z) \cos(\theta) \quad (2.10)$$

In the Equations (2.8) the first subscript  $l = D, 1, 3, 5$  denotes the respective boundary value problem while the second one denotes the values of  $m$  that should be taken into account in the solution process. Therefore, the functions  $\Psi_{lm}^k$ ,  $k = I, II, III$  remain the main unknown of the problem.

The complex velocity potentials  $\phi_j$ ,  $j = 0, 1, 3, 5, 7$  have to satisfy:

The Laplace equation in the entire corresponding fluid domain:

$$\Delta\phi_j^k = \frac{\partial^2\phi_j^k}{\partial r^2} + \frac{1}{r}\frac{\partial\phi_j^k}{\partial r} + \frac{1}{r^2}\frac{\partial^2\phi_j^k}{\partial\theta^2} + \frac{\partial^2\phi_j^k}{\partial z^2}, \quad k = I, II, III \quad (2.11)$$

The free surface boundary condition:

$$-\omega^2\phi_j^k + g\frac{\partial\phi_j^k}{\partial z} = 0, \quad z = d; \quad k = I, II \quad (2.12)$$

And the impermeable boundary condition at the seabed:

$$\frac{\partial\phi_j^k}{\partial z} = 0, \quad z = 0; \quad k = I, III \quad (2.13)$$

In addition, the radiation and scattering potentials  $\phi_j$ ,  $j = 1, 3, 5, 7$  have to fulfil the Sommerfeld radiation condition at the far-field [50]:

$$\lim_{r \rightarrow \infty} \sqrt{kr} \left( \frac{\partial\phi_j^k}{\partial r} - ik\phi_j^k \right) = 0 \quad (2.14)$$

Furthermore, the kinematic condition at the impermeable wetted surface  $S$  of the body, in its average position, has to be satisfied:

$$\frac{\partial\phi_7^k}{\partial n} = -\frac{\partial\phi_0}{\partial n} = 0, \quad \text{on } S \quad (2.15)$$

and,

$$\frac{\partial\phi_j^k}{\partial n} = n_j, \quad \text{on } S; \quad j = 1, 3, 5 \quad (2.16)$$

In Equations (2.15) and (2.16)  $\frac{\partial}{\partial n}$  denotes the derivative in the direction of the outward unit normal vector,  $n$ , to the mean wetted surface of the body and  $n_j$  is the generalised normal vector defined as:

$$(n_1, n_2, n_3) = n \quad (2.17)$$

and

$$(n_4, n_5, n_6) = r \times n \quad (2.18)$$

where  $r$  is the position vector with respect to the origin of the coordinate system.

The flow through the porous openings of the body's sidewall is characterized by relatively small Reynold's numbers, so it can be assumed to obey the linear Darcy's law [51]. The normal flow velocity is assumed to be continuous and linearly proportional to the pressure difference through the porous boundary [25]. Hence, the boundary condition on the porous surface of the body forms:

$$\frac{\partial\phi_j^{II}}{\partial r} = n_j + ikG(\phi_{II} - \phi_I) \quad \text{on } r = a, \quad h_1 \leq z \leq d, \quad j = 1, 3, 5, D \quad (2.19)$$

where  $G$  denotes the complex dimensionless porous effect parameter. This coefficient can be composed as  $G = G_{re} + iG_{im}$ , where  $G_{re}$  is its real part representing the linearised drag effect

on the porous sidewall and  $G_{im}$  its imaginary part, representing the inertial effect. Therefore, for a real  $G$ , the resistance effects dominate over inertia effects, while  $G$  acquires complex values when inertia effects dominate over resistance ones. The parameter  $G$  also expresses a measure for the porosity effect, when its value tends to zero  $G = 0$  the sidewall is completely impermeable, while when it tends to values much higher than unity,  $G \gg 1$ , the sidewall is completely permeable, i.e. there is no sidewall. In [1], it was proven experimentally that the porous coefficient  $G$  is related to the opening rate  $\tau$  of the material, the ratio between the area of the opening holes and the total area, as well as the waveslope  $\epsilon = \frac{kH}{2}$ , through the following Equation:

$$G = \frac{(\frac{17.8}{\epsilon} + 432.2)\tau^2}{2\pi(1 + 1.06\tau)} \quad (2.20)$$

As the flow is considered to be continuous on either side of the boundary, both the velocity potential and its derivative must be continuous at the vertical boundaries of adjacent fluid regions.

- at  $r = a$  for  $0 \leq z \leq h$

$$\phi_j^I = \phi_j^{III} \quad (2.21)$$

$$\frac{\partial \phi_j^I}{\partial r} = \frac{\partial \phi_j^{III}}{\partial r} \quad (2.22)$$

- at  $r = a$  for  $h_1 \leq z \leq d$

$$\frac{\partial \phi_j^I}{\partial r} = \frac{\partial \phi_j^{II}}{\partial r} \quad (2.23)$$

## 2.2 Analytical Solutions

- Infinite flow region  $I$

$$\frac{1}{\delta_j} \Psi_{lm}^I(r, z) = f_{lm}^I(r, z) + \sum_{i=0}^{\infty} F_{lm,i}^I \frac{K_m(a_i^I r)}{K_m(a_i^I a)} Z_i(z) \quad (2.24)$$

where  $K_m$  is the  $m$ -th order modified Bessel function of the second kind and  $\delta_l = d$ , for  $l = D, 1, 3$  and  $\delta_5 = d^2$ . Whereas, the term  $f_{lm}^I$  equals to:

$$f_{11}^I = f_{30}^I = f_{50}^I = 0 \quad (2.25)$$

and

$$f_{Dm}^I = (J_m(kr) - \frac{J_m(ka)}{H_m(ka)} H_m(kr)) \frac{Z_0(z)}{dZ_0(z)} \quad (2.26)$$

where the term  $H_m$  is the  $m$ -th order Hankel function of the first kind.

The terms  $Z_i(z)$ , in Equations (2.24) and (2.45), are the orthonormal functions in  $[0, d]$  defined as follows:

$$Z_0(z) = N_{0,I}^{-\frac{1}{2}} \cosh(k^I z) \quad (2.27)$$

$$Z_i(z) = N_{i,I}^{-\frac{1}{2}} \cosh(a_i^I z) \quad (2.28)$$

where

$$N_{0,I} = \frac{1}{2} \left[ 1 + \frac{\sinh(2k^I d)}{2k^I d} \right] \quad (2.29)$$

$$N_{i,I} = \frac{1}{2} \left[ 1 + \frac{\sinh(2a_i^I d)}{2a_i^I d} \right] \quad (2.30)$$

The wave number  $k^I \equiv k$  and is related to  $\omega$  through the dispersion relation (Equation (2.6)).

The terms  $a_i^I$  are the positive real roots of the transcendental equation:

$$\frac{\omega^2}{g} + a_i^I \tan(a_i^I d) = 0 \quad (2.31)$$

and the imaginary root  $a_0^I = -ik$ .

- Second flow region *II*

$$\frac{1}{\delta_j} \Psi_{lm}^{II}(r, z) = f_{lm}^{II}(r, z) + \sum_{i=0}^{\infty} F_{lm,i}^{II} \frac{I_m(a_i^{II} r)}{I_m(a_i^{II} a)} Z_i(z - h_1) \quad (2.32)$$

where the term  $I_m$  is the m-th order modified Bessel function of the first kind and the term  $f_{lm}^{II}$  is equal to:

$$f_{Dm}^{II} = f_{11}^{II} = 0 \quad (2.33)$$

$$f_{30}^{II} = \frac{z}{d} - 1 + \frac{g}{d\omega^2} \quad (2.34)$$

$$f_{51}^{II} = -\frac{r}{d^2} \left( (z - d) + \frac{g}{\omega^2} \right) \quad (2.35)$$

Where the term  $Z_i(z - h_1)$  is derived as follows:

$$Z_0(z - h_1) = N_{0,II}^{-\frac{1}{2}} \cosh(k^{II}(z - h_1)) \quad (2.36)$$

$$Z_i(z - h_1) = N_{i,II}^{-\frac{1}{2}} \cosh(a_i^{II}(z - h_1)) \quad (2.37)$$

where

$$N_{0,II} = \frac{1}{2} \left[ 1 + \frac{\sinh(2k^{II}(d - h_1))}{2k^{II}(d - h_1)} \right] \quad (2.38)$$

$$N_{i,II} = \frac{1}{2} \left[ 1 + \frac{\sinh(2a_i^{II}(d - h_1))}{2a_i^{II}(d - h_1)} \right] \quad (2.39)$$

The terms  $a_i^{II}$  are the roots of the transcendental equation:

$$\frac{\omega^2}{g} + a_i^{II} \tan(a_i^{II}(d - h_1)) = 0 \quad (2.40)$$

with the imaginary root  $a_i^{II} = -ik^{II}$  to be considered as first.

- Third flow region *III*

$$\frac{1}{\delta_j} \Psi_{lm}^{III}(r, z) = f_{lm}^{III}(r, z) + \sum_{n=0}^{\infty} \epsilon_n F_{lm,n}^{III} \frac{I_m\left(\frac{n\pi}{h} r\right)}{I_m\left(\frac{n\pi}{h} a\right)} \cos\left(\frac{n\pi}{h} z\right) \quad (2.41)$$

where the term  $f_{lm}^{III}$  equals to:

$$f_{Dm}^{III} = f_{II}^{III} = 0 \quad (2.42)$$

$$f_{30}^{III} = \frac{z^2 - 0.5r^2}{2hd} \quad (2.43)$$

$$f_{51}^{III} = \frac{-r(z^2 - 0.25r^2)}{2hd^2} \quad (2.44)$$



## 2.3 Hydrodynamic Forces

The various hydrodynamic forces acting on the permeable body are calculated by the pressure distribution given by the linearized Bernoulli's equation. Therefore they can be written as:

$$F_1 = -2\pi i \rho \omega^2 d \frac{H}{2} a \left[ \int_h^d \frac{1}{d} \Psi_{D1}^I(a, z) dz - \int_{h1}^d \frac{1}{d} \Psi_{D1}^{II}(a, z) dz \right] \quad (2.45)$$

Given the Equations (2.24) and (2.32)), Equation (2.45) becomes:

$$\begin{aligned} \frac{F_1}{B} = & -2ikd \tanh(kd) \left[ N_{0,I}^{-\frac{1}{2}} F_{D1,0}^I \frac{\sinh(kd) - \sinh(kh)}{ka} + \sum_{i=1}^{\infty} N_{i,I}^{-\frac{1}{2}} F_{D1,i}^I \frac{\sin(a_i^I d) - \sin(a_i^I h)}{a_i^I a} \right] \\ & - N_{0,II}^{-\frac{1}{2}} F_{D1,0}^{II} \frac{\sinh(k^{II}(d - h_1))}{k^{II} a} - \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{D1,i}^{II} \frac{\sin(a_i^{II}(d - h_1))}{a_i^{II} a} \end{aligned} \quad (2.46)$$

where  $B = \pi \rho g a^2 (\frac{H}{2})$ ,  $\rho$  is the water density of seawater,  $g$  is the gravity acceleration. Accordingly, the vertical forces acting on the permeable body are equal to the sum of the forces acting on the upper surface ( $0 \leq r \leq a$ ;  $z = h_1$ ) and lower surface ( $0 \leq r \leq a$ ;  $z = h$ ). Therefore,

$$F_3 = 2\pi \rho \omega^2 d \frac{H}{2} \left[ \int_0^a \frac{1}{d} \Psi_{D0}^{III}(r, h) r dr - \int_0^a \frac{1}{d} \Psi_{D0}^{II}(r, h_1) r dr \right] \quad (2.47)$$

When substituting Equations (2.32) and (2.41) in Equation (3.12b), we get

$$\begin{aligned} \frac{F_3}{B} = & 2kd \tanh(kd) \left[ \frac{1}{2} F_{D0,0}^{III} + 2 \sum_{i=1}^{\infty} (-1)^n F_{D0,i}^{III} \frac{1}{h} \frac{I_1(\frac{n\pi a}{h})}{I_0(\frac{n\pi a}{h})} \right] \\ & - N_{0,II}^{-\frac{1}{2}} F_{D0,0}^{II} \frac{1}{k^{II} a} \frac{J_1(k^{II} a)}{J_0(k^{II} a)} - \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{D0,i}^{II} \frac{1}{a_i^{II}} \frac{I_1(a_i^{II} a)}{I_0(a_i^{II} a)} \end{aligned} \quad (2.48)$$

The moment on the permeable body about a horizontal axis in arbitrary distance  $z = e$  from the seabed can be analysed in terms of  $M_S$  and  $M_b$  obtained from the distribution of pressure on the wetted surfaces of the body, both permeable and impermeable, and it holds:

$$M_S = -2i\pi \rho \omega^2 d \frac{H}{2} a \left[ \int_h^d \frac{1}{d} \Psi_{D1}^I(a, z) (z - e) dz - \int_{h1}^d \frac{1}{d} \Psi_{D1}^{II}(a, z) (z - e) dz \right] \quad (2.49)$$

$$M_b = -2\pi \rho \omega^2 d \frac{H}{2} \left[ \int_0^a \frac{1}{d} \Psi_{D1}^{III}(r, h) r^2 dr - \int_0^a \frac{1}{d} \Psi_{D1}^{II}(r, h_1) r^2 dr \right] \quad (2.50)$$

Substituting Equations (2.24), (2.32), and (2.41) into Equations (2.49) and (2.50), it is derived that

$$\frac{M_S}{Ba} = -2ikd \tanh(kd) \left[ -N_{0,I}^{-\frac{1}{2}} F_{D1,0}^I L_0^I + \sum_{i=1}^{\infty} N_{i,I}^{-\frac{1}{2}} F_{D1,i}^I L_i^I - N_{0,II}^{-\frac{1}{2}} F_{D1,0}^{II} L_0^{II} - \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{D1,i}^{II} L_i^{II} \right] \quad (2.51)$$

$$\begin{aligned} \frac{M_b}{Ba} = & -2kd \tanh(kd) \left[ \frac{1}{4} F_{D1,0}^{III} + 2 \sum_{i=1}^{\infty} (-1)^n F_{D1,i}^{III} \frac{1}{h} \frac{I_2\left(\frac{n\pi}{h}a\right)}{I_1\left(\frac{n\pi}{h}a\right)} \right. \\ & \left. - N_{0,II}^{-\frac{1}{2}} F_{D1,0}^{II} \frac{1}{k^{II}a} \frac{J_2(k^{II}a)}{J_1(k^{II}a)} - \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{D1,i}^{II} \frac{1}{a_i^{II}} \frac{I_2(a_i^{II}a)}{I_1(a_i^{II}a)} \right] \end{aligned} \quad (2.52)$$

where the terms  $L_0^I, L_0^{II}, L_i^I, L_i^{II}$  are defined as follows,

$$L_0^I = \frac{k(d-e) \sinh(kd) - k(h-e) \sinh(kh) - \cosh(kd) + \cosh(kh)}{(ka)^2} \quad (2.53)$$

$$L_i^I = \frac{a_i^I(d-e) \sin(a_i^I d) - a_i^I(h-e) \sin(a_i^I h) + \cos(a_i^I d) - \cos(a_i^I h)}{(a_i^I a)^2} \quad (2.54)$$

$$L_0^{II} = \frac{-1 - k^{II}(d-e) \sinh(k^{II}(d-h_1)) + \cosh(k^{II}(d-h_1))}{(k^{II}a)^2} \quad (2.55)$$

$$L_i^{II} = \frac{-1 + a_i^{II}(d-e) \sin(a_i^{II}(d-h_1)) + \cos(a_i^{II}(d-h_1))}{(a_i^{II}a)^2} \quad (2.56)$$

Similarly due to the permeable body's sinusoidal motion in the the  $i$ -th direction with frequency  $\omega$  and unit amplitude in the  $j$ -th direction, the corresponding reaction forces and pitching moment  $F_{ij}$  equal to:

$$F_{ij} = -\rho\omega^2 \iint_S \Psi_{j0}^y(r, z) \cos(m\theta) n_i dS, \quad y = I, II, III; \quad i, j = 1, 3, 5 \quad (2.57)$$

where  $S$  stands for the wetted surface and  $n_i$  as defined in Equation (2.16). Furthermore, Equation (2.57) can also be written as:

$$F_{ij} = \omega^2 a_{ij} + i\omega b_{ij} \quad (2.58)$$

where  $a_{ij}$  and  $b_{ij}$  denote the hydrodynamic mass and damping coefficients in the  $i$ -th direction due to the unit sinusoidal motion of the body in the  $j$ -th direction. It is worth noted that both  $a_{ij}$  and  $b_{ij}$  are real and  $\omega$ -dependent.

Substituting Equations (2.24), (2.32) and (2.41) into equation (2.58) gives the following relationships for the non-dimensional hydrodynamic coefficients:

$$\begin{aligned} \frac{a_{11}}{\rho a^3} + i \frac{b_{11}}{\rho \omega a^3} = & -\pi \frac{d}{a} \left[ N_{0,I}^{-\frac{1}{2}} F_{11,0}^I \frac{\sinh(kd) - \sinh(kh)}{ka} + \sum_{i=1}^{\infty} N_{i,I}^{-\frac{1}{2}} F_{11,i}^I \frac{\sin(a_i^I d) - \sin(a_i^I h)}{a_i^I a} \right. \\ & \left. - N_{0,II}^{-\frac{1}{2}} F_{11,0}^{II} \frac{\sinh(k^{II}(d-h_1))}{k^{II}a} - \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{11,i}^{II} \frac{\sin(a_i^{II}(d-h_1))}{a_i^{II}a} \right] \end{aligned} \quad (2.59)$$

$$\begin{aligned} \frac{a_{33}}{\rho a^3} + i \frac{b_{33}}{\rho \omega a^3} = & 2\pi \frac{d}{a} \left[ \left( \frac{h}{4d} - \frac{a^2}{16hd} + \frac{1}{2} F_{33,0}^{III} + 2 \sum_{n=1}^{\infty} (-1)^n F_{33,i}^{III} \frac{1}{h} \frac{I_1\left(\frac{n\pi}{h}a\right)}{I_0\left(\frac{n\pi}{h}a\right)} \right) \right. \\ & \left. - \frac{1}{2} \left( \frac{h_1}{d} - 1 + \frac{g}{\omega^2 d} \right) - N_{0,II}^{-\frac{1}{2}} F_{30,0}^{II} \frac{1}{k^{II}a} \frac{J_1(k^{II}a)}{J_0(k^{II}a)} - \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{30,i}^{II} \frac{1}{a_i^{II}a} \frac{I_1(a_i^{II}a)}{I_0(a_i^{II}a)} \right] \end{aligned} \quad (2.60)$$

$$\frac{a_{55}}{\rho a^5} + i \frac{b_{55}}{\rho \omega a^5} = -\pi \frac{d^2}{a^2} (M_S + M_B) \quad (2.61)$$

The terms  $M_S$  and  $M_B$  are defined as follows:

$$M_S = N_{0,I}^{-\frac{1}{2}} F_{51,0}^I L_0^I + \sum_{i=1}^{\infty} N_{i,I}^{-\frac{1}{2}} F_{51,i}^I L_i^I - N_{0,II}^{-\frac{1}{2}} F_{51,0}^{II} L_0^{II} + \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{51,i}^{II} L_i^{II} \quad (2.62)$$

$$M_B = \frac{1}{8} \frac{ah}{d^2} - \frac{1}{48} \frac{a^3}{d^2 h} + \frac{1}{4} F_{51,0}^{III} + 2 \sum_{n=1}^{\infty} (-1)^n F_{51,i}^{III} \frac{1}{h} \frac{I_2(\frac{n\pi a}{h})}{I_1(\frac{n\pi a}{h})} - \frac{ag}{4d^2 \omega} + \frac{a(d-h_1)}{4d^2} \quad (2.63)$$

$$- N_{0,II}^{-\frac{1}{2}} F_{51,0}^{II} \frac{1}{k^{II} a} \frac{J_2(k^{II} a)}{J_1(k^{II} a)} - \sum_{i=1}^{\infty} N_{i,II}^{-\frac{1}{2}} F_{51,i}^{II} \frac{1}{a_i^{II} a} \frac{I_2(a_i^{II} a)}{I_1(a_i^{II} a)}$$

The terms  $L_0^I, L_i^I, L_0^{II}, L_i^{II}$  are defined in Equations (2.53)-(2.56).

## 2.4 Equations of Motions

In the present study, the assumption of small motions and rotations in the free floating body's six degrees of freedom is made. This allows the the motions and rotation of the body to be calculated as follows:

$$\sum_{j=1}^6 [-\omega^2(m_{ij} + a_{ij}) - i\omega b_{ij} + c_{ij}] x_{j0} = F_i \quad (2.64)$$

The terms  $m_{ij}$  in Equation 2.64 denote the generalized masses of the floating body,  $a_{ij}$  and  $b_{ij}$  the hydrodynamic added mass and damping coefficients respectively,  $c_{ij}$  the hydrostatic stiffness coefficients and  $F_i$  the exciting forces and moments.

The free floating permeable cylinder body is assumed to have symmetrical distribution of mass and to perform two translations moves, more precisely surge and heave, and one rotational move, pitch. Following these assumptions, the motions can be expressed in terms of response amplitude operator (RAOs) as follows:

$$\text{RAO}_i = \frac{x_{j0}}{H/2}, \quad i=1,3 \quad (2.65)$$

and

$$\text{RAO}_5 = \frac{x_{j0}}{kH/2}, \quad i=5, \quad (2.66)$$

where  $k$  denotes the wave number.

## 2.5 Response Spectrum

The response spectrum results from the transformation of wave energy into response energy of a floating body in the frequency domain. The principle of this transformation is shown in Figure [2.3]. for the heave motions being considered here.

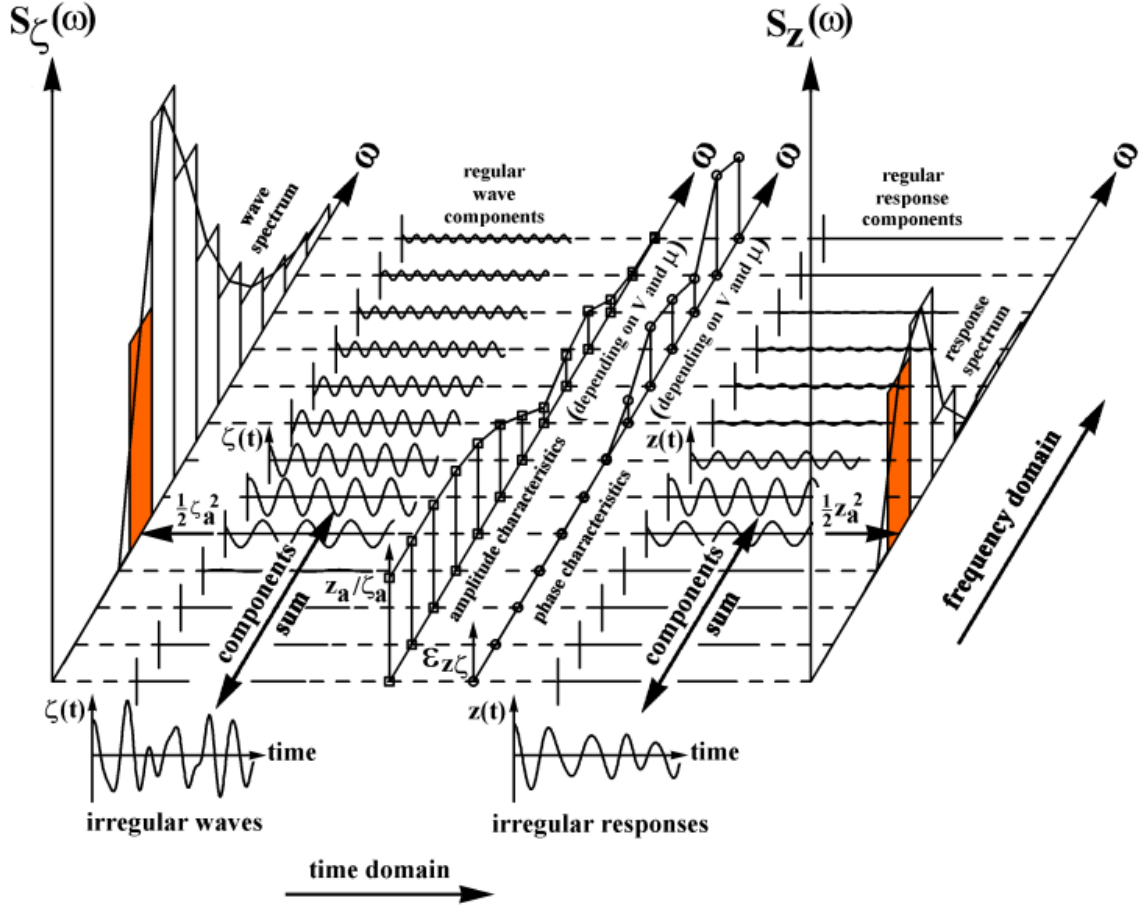


Figure 2.2: Principle of Transfer of Waves into Responses [52]

In Figure [2.3] the wave spectrum is depicted by  $S_\zeta(\omega)$  and the term  $\zeta(t)$  denotes the irregular wave history, which in terms of the linear wave theory, is the sum of regular waves components, each one with its own frequency, amplitude and energy. Figure [2.3] also shows the the response spectra for heave motion, denoted as  $S_z(\omega)$ . The regular heave components are obtained by multiplying each of the wave components by the motion RAO and then superpositioned to obtain the irregular heave history,  $z(t)$ .

### 2.5.1 JONSWAP Spectrum

It is known that, the wave spectrum describes the distribution of energy between wave components of different frequencies of a sea state. Wave spectra can be obtained directly from measured data. However, various mathematical formulas of wave spectra are available based on analysis of measured data, most notably, for fetch-limited regions, the Joint North Sea Wave Project (JONSWAP). The spectral density of the JONSWAP spectrum is described by [53]:

$$S_\zeta(\omega) = A_\gamma S_{PM}(\omega) \gamma^{\exp[-\frac{1}{2}(\frac{\omega-\omega_p}{\sigma\omega_p})^2]} \quad (2.67)$$

where,

- $A_\gamma = 1 - 0.287 \ln(\gamma)$
- $\gamma = 3.3$  peakedness factor

- $S_{PM}(\omega) = \frac{5}{16}(H_s^2 \omega_p^4) \omega^{-5} \exp(-\frac{5}{4}(\frac{\omega}{\omega_p})^{-4})$
- $\sigma = \begin{cases} 0.07 & (\omega \leq \omega_p) \\ 0.09 & (\omega \geq \omega_p) \end{cases}$

where  $\omega$  and  $\omega_p$  are the the incident wave frequency and modal frequency, respectively.  $A_\gamma$  is the normalising factor with  $\gamma$  referring to the non-dimensional peak-enhancement factor.  $S_{PM}(\omega)$  is the Pierson-Moskowitz spectrum,  $H_s$  is the significant wave height, and  $\sigma$  denotes the spectral width parameter as shown in Figure [2.3].

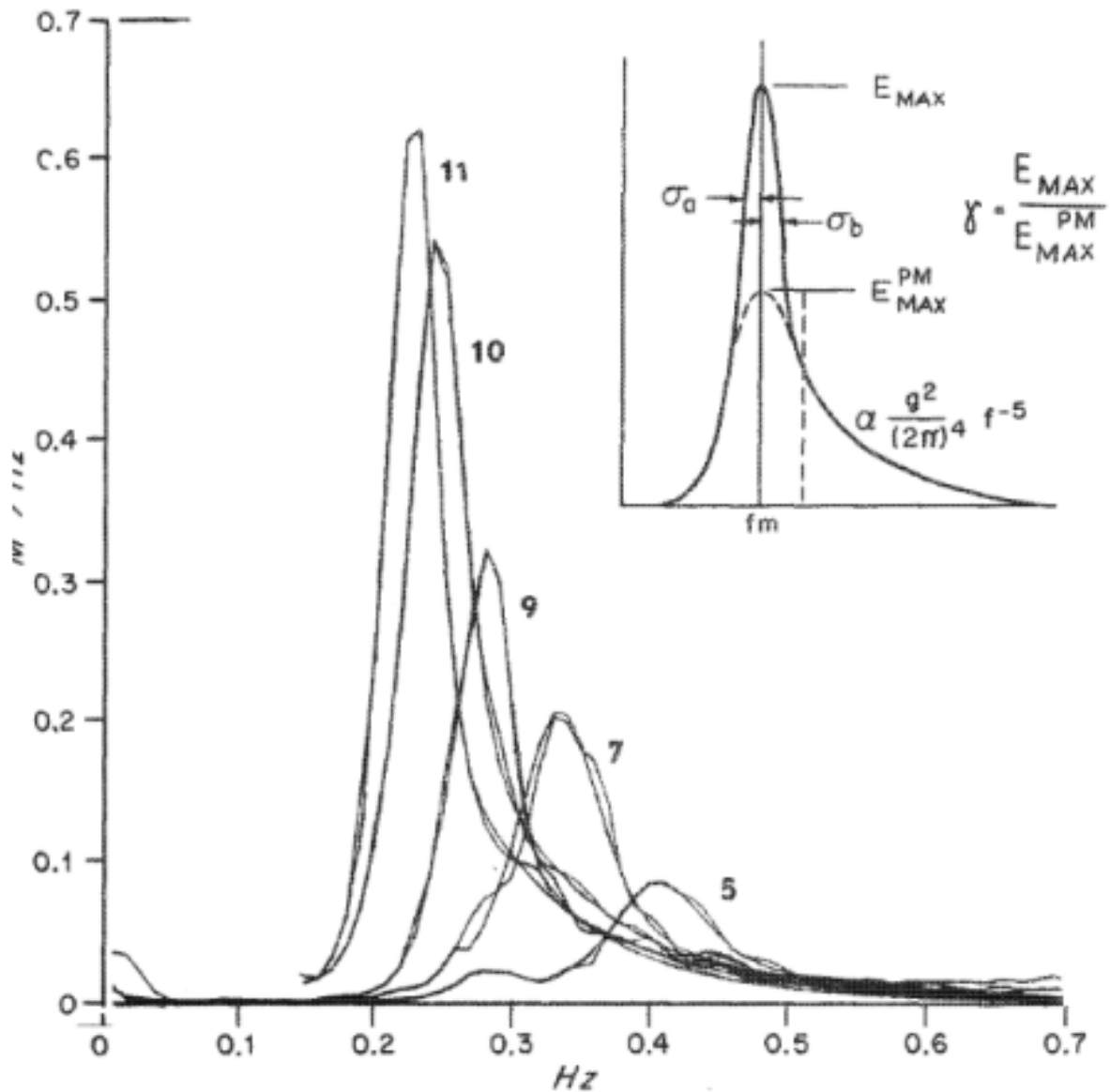


Figure 2.3: Measured wave spectra under ideal conditions and parametrical approximations. Principal sketch of the peak shape parameters  $\gamma$ ,  $\sigma_a$  and  $\sigma_b$ . [53]

### 2.5.2 Climate Data

In this study, various values of climate data, significant wave height  $H_s$  and spectral period  $T_p$  were used to calculate the response spectrum of the examined body correspond. The upper limit of those values was determined by the maximum significant wave height  $H_s$  and the corresponding spectral wave period  $T_p$  as calculated in [54], for a given location in Greek seas.

$H_s$ (m)	5.48
$T_p$ (s)	10.28

Table 1: Climate Data

### 3 Results

#### 3.1 Validation

In order to validate the numerical method used in the present study, its results are compared to corresponding ones available in the literature. The following results are produced by code developed in FORTRAN. The examined body is a porous free floating cylinder with a radius of  $a$ , draught ( $d - h_1/a = 2$ ) that has an impermeable bottom, of negligible thickness and is subjected to waves propagating at zero angle in a constant water depth of  $d = 100a/3$ . The porous coefficient  $G$  is assumed to be a real number and equal to  $G=1.432$ . As can be seen, in Figures 1 and 2 the hydrodynamic coefficients and exciting forces respectively are in total accordance to the results reported in [32]. In order for them to be comparable, the exciting forces and moment have been normalized by  $\rho g a^2/(H/2)$  and  $\rho g a^3/(H/2)$  while the hydrodynamic added mass and damping by  $\rho a^3$  and  $\rho \omega a^3$  respectively.

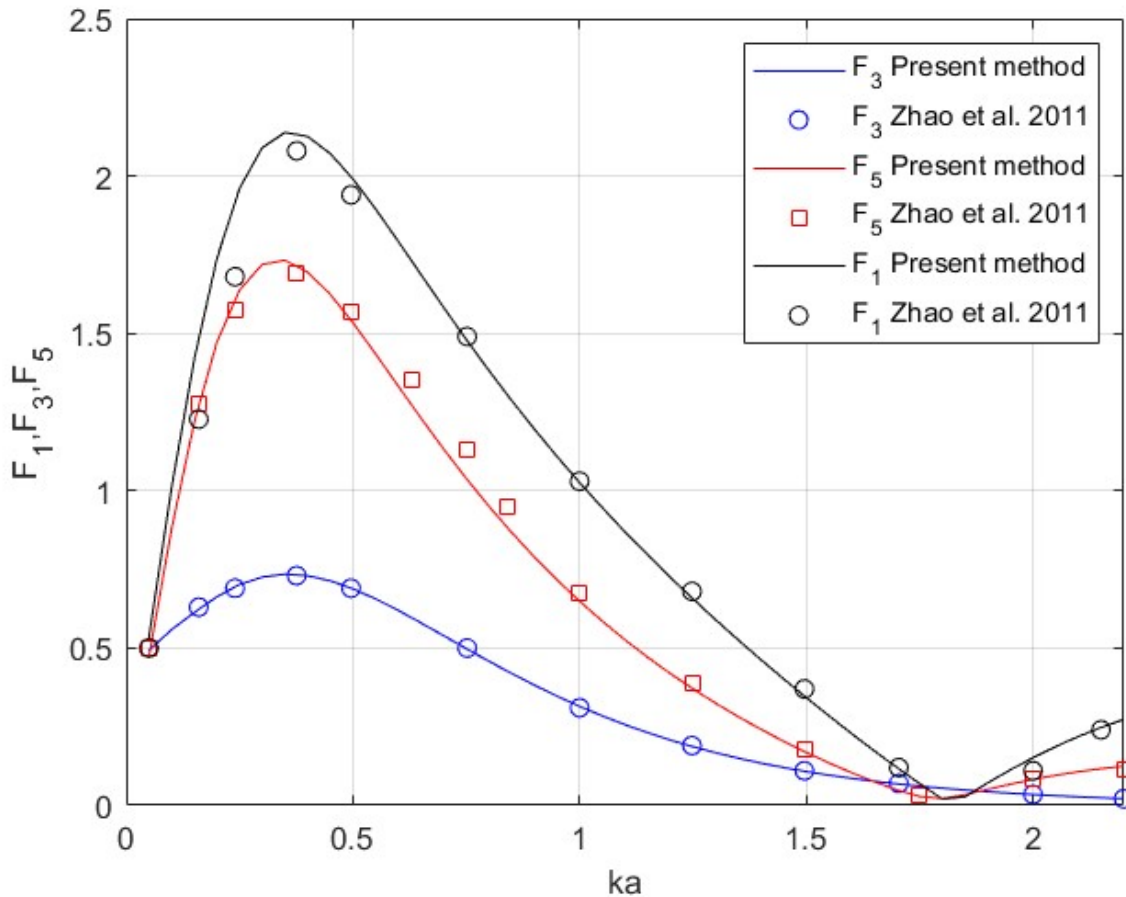


Figure 3.1: Comparison of the present method results against those from Zhao et al. the exciting forces and moments on a porous cylinder

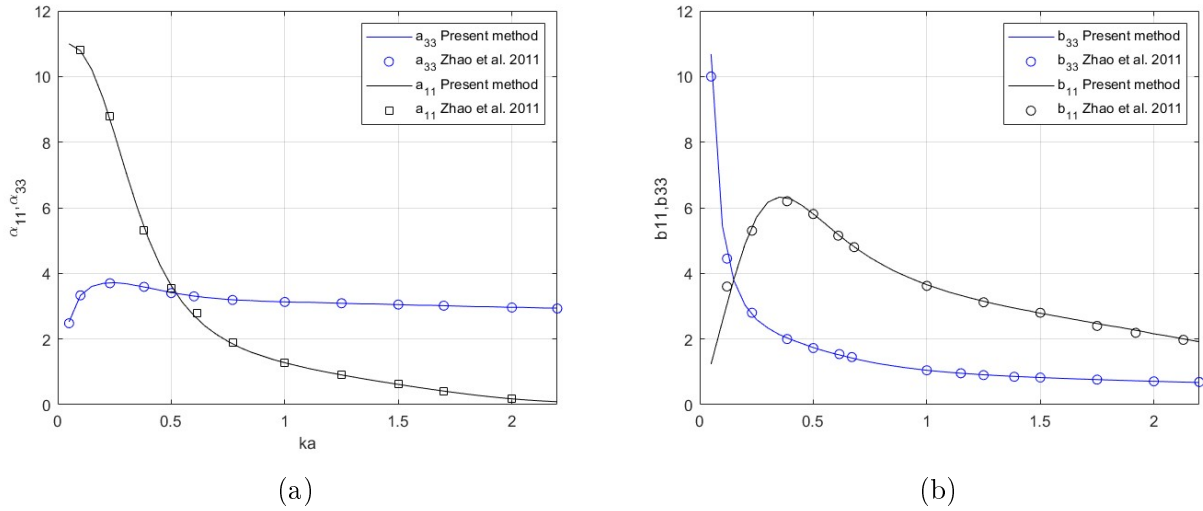


Figure 3.2: Comparison of the present method results against those due to Zhao et al. [32], concerning the hydrodynamic added mass in surge and heave (3.2a), hydrodynamic damping coefficients in surge and heave (3.2b)

## 3.2 Test Cases

### 3.2.1 Effect of the Porous G coefficient

In this section the effect of the porous coefficient in the exciting forces and hydrodynamic coefficients is explored. The examined body is, as aforementioned, a porous free floating cylinder with a radius of  $a = 0.15$ , draught  $(d - h_1)/a = 2$  that has an impermeable bottom, of negligible thickness and is subjected to waves propagating at zero angle in a constant water depth of  $d = 100\alpha/3$ . The body's mass is assumed to be constant regardless of the porosity and equal to  $M = 3.62 * 10^{-4} t$ , the center of gravity is located at the body's vertical axis at  $4.697 m$  below the free surface and the mass moment of inertia relative to the free surface equals to  $I_z = 4.075 * 10^{-6} tm^2$

In Table 1 (2) the values of opening ratios  $\tau$  and the corresponding values of  $G$ , as derived from the Equation (2.20), are depicted. The wave slope is considered to be  $\epsilon = 0.05$  [36]. Furthermore, the cases of  $G = 0$  and  $G \gg 1$ , the body being fully impermeable and fully permeable respectively, are considered.

$\tau$	0.037	0.08	0.12	0.14	0.22	0.41	0.60
$G$	0.101	0.468	1.015	1.432	3.118	9.309	17.482

Table 2: Opening ratios  $\tau$  and the corresponding porous coefficient  $G$  for wave slope  $\epsilon = 0.05$

The effect the porous coefficient  $G$  has on the surge, heave and pitch exciting forces and moment is depicted in Figure [3.3], while the effect on the corresponding hydrodynamic coefficients of added mass and damping is depicted in Figures [3.4]-[3.5].



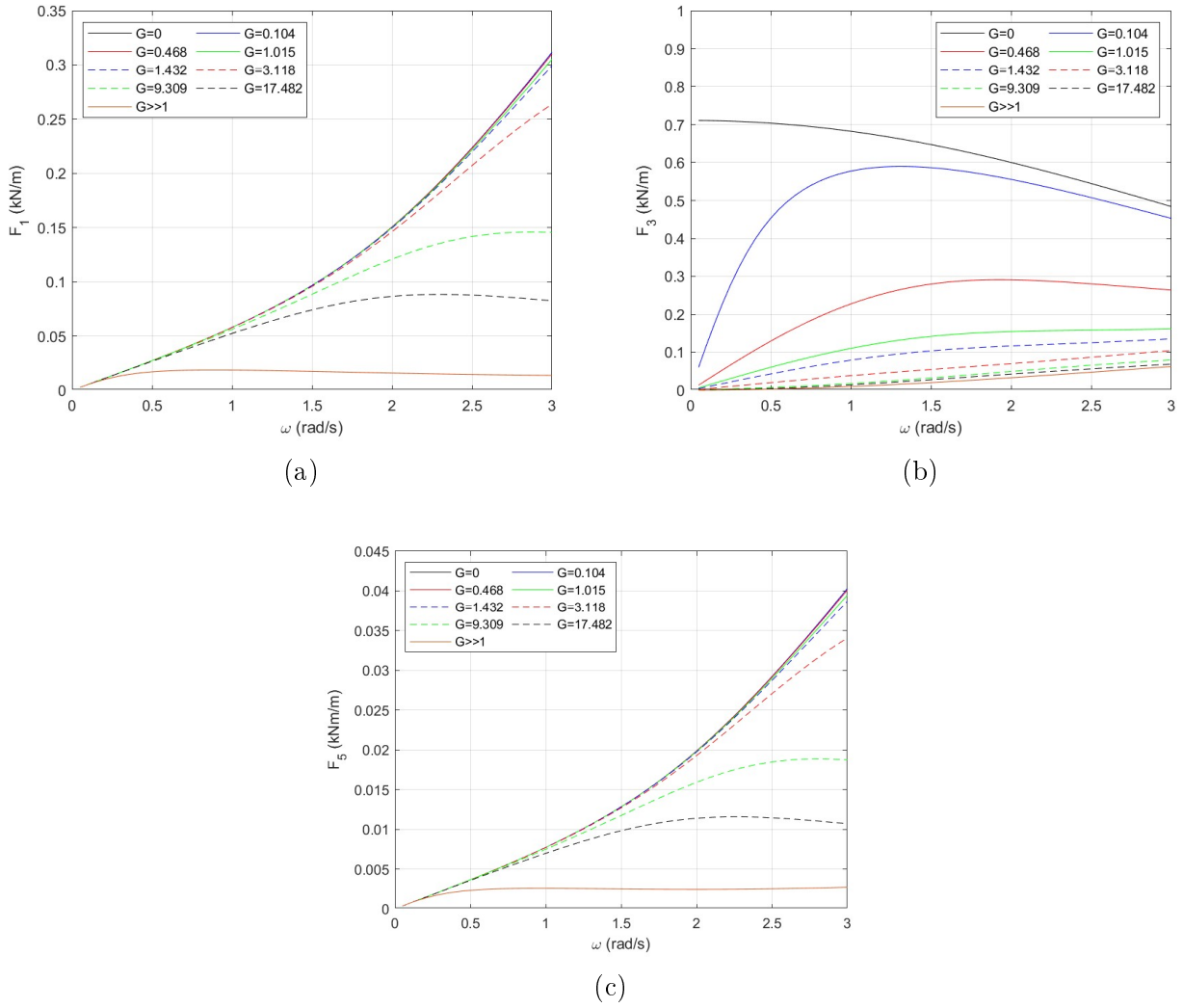


Figure 3.3: Exciting forces  $F_1$ ,  $F_3$  and moment  $F_5$  on the permeable cylindrical body against wave frequency  $\omega$  for various values of porous effect parameter

In Figure [3.3b], it can be clearly seen that the heave exciting forces behave differently for  $G = 0$  and  $G > 0$ . For the case of impermeable cylinder,  $G = 0$ , the heave exciting force has a value of about  $0.7 \text{ kN/m}$  for  $\omega \simeq 0$  whereas in the same frequencies, for cases of  $G > 0$ ,  $F_3$  tends to zero as  $G$  increases. This pattern, the decline of  $F_3$  as  $G$  increases, is also observed at higher frequencies. It is also worth noted, that as  $G$  increases the local maxima of  $F_3$  are observed in higher frequencies. As far as the surge excitation force  $F_1$ , Figure [3.3a], and the overturning moment  $F_5$ , Figure [3.3c], are concerned, the effect of  $G$  is not significant in the lower frequency range  $\omega < 1 \text{ rad/s}$ . However, as the frequencies become higher, it becomes clearer that again, as the value of  $G$  increases, the excitation force and the overturning moment decrease. This holds true for all cases of  $G$  except of the fully submerged plate,  $G \gg 1$  where both  $F_1$  and  $F_5$  have a constant value for frequencies greater than  $\omega = 0.2 \text{ rad/s}$ .

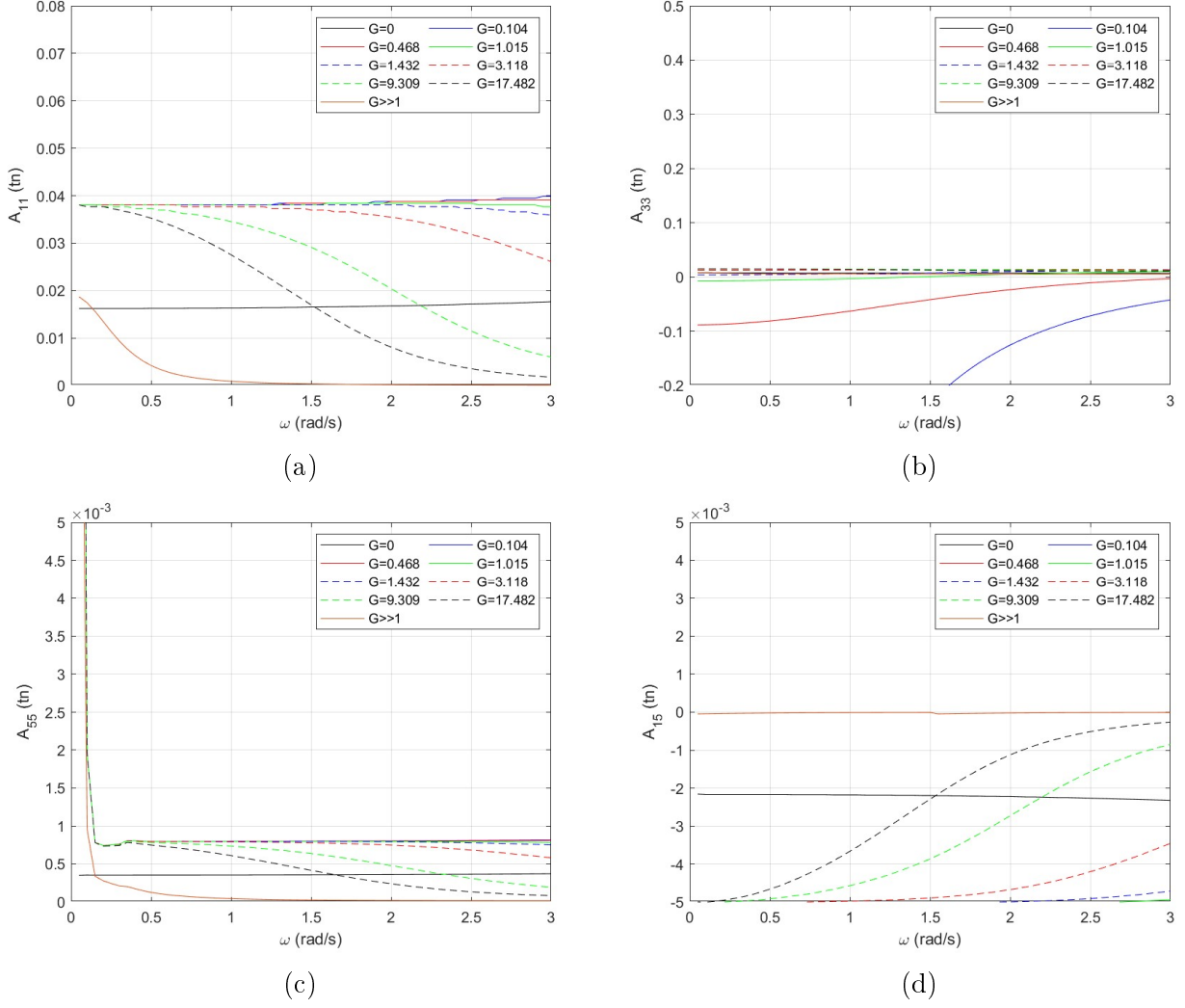


Figure 3.4: Hydrodynamic added mass  $A_{11}$   $A_{33}$ ,  $A_{55}$   $A_{15}$  of the permeable cylindrical body against wave frequency  $\omega$  for various values of porous effect parameter.

Regarding the added mass in the wave direction, as depicted in figure [3.4a], it is noted that  $A_{11}$  decreases as  $G$  increases and gradually tends to zero in the higher frequency range. This trend is more evident in the case of higher values of  $G$  and especially in the case of the fully submerged plate,  $G \gg 1$ , which exhibits a steeper decline towards zero. As far as, heave added mass,  $A_{33}$ , is concerned, it can be seen in Figure [3.4b] that for low frequencies, the added mass behaves proportionally to  $G$ , meaning it increases as  $G$  does, tending to the values of the submerged plate case  $G \gg 1$ . It is worth noted that the pitch added mass  $A_{55}$ , as can be observed in Figure [3.4c], is characterized by the existence of local maxima at the lower frequency range  $\omega < 0.2 \text{ rad/s}$  for all cases of  $G$ , expect the one of impermeable cylinder  $G = 0$ . At higher frequencies though, the same pattern as in Figure [3.4a] is observed. Similarly, the surge pitch added mass, Figure [3.4d], follows a similar rational, for greater values of  $G$  smaller values of  $A_{15}$  are noted. It is also noted that due to the body's symmetry  $A_{15} = A_{51}$  which also holds true for this study.

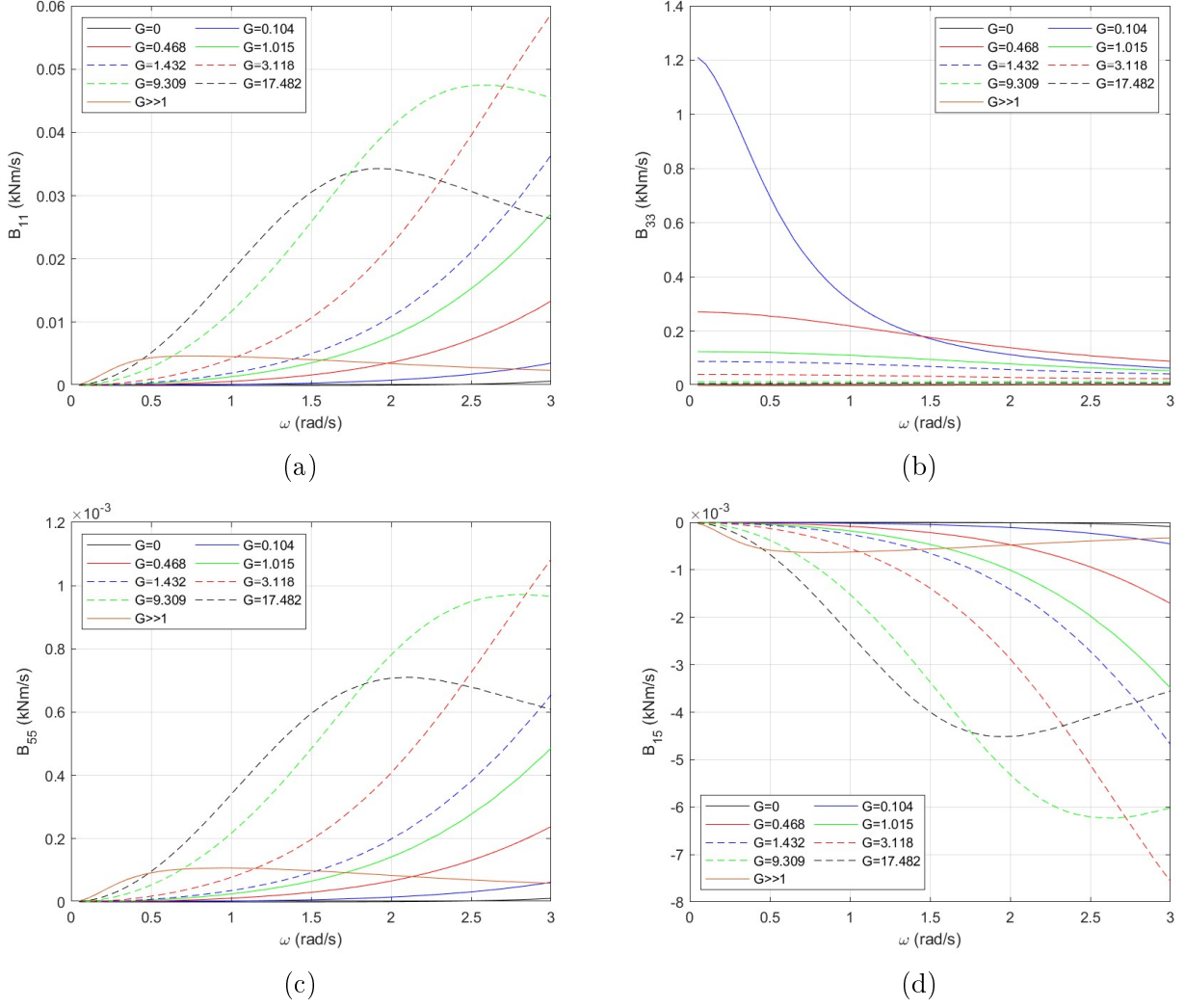


Figure 3.5: Hydrodynamic damping  $B_{11}$   $B_{33}$ ,  $B_{55}$   $B_{15}$  of the permeable cylindrical body against wave frequency  $\omega$  for various values of porous effect parameter.

In Figure [3.5] the effect the porous coefficient has on the hydrodynamic damping of the porous cylindrical body, is shown. More specifically, in Figure [3.5a] one can notice that for lower frequencies  $\omega < 0.5 \text{ rad/s}$ ,  $B_{11}$  behaves proportionally, as the hydrodynamic damping increases as  $G$  does. However, for higher frequencies  $\omega > 1.5 \text{ rad/s}$  the behaviour reverses. Now, for higher values of  $G$  the local maxima of  $B_{11}$  are shifted to higher values. The same holds true for the cases of pitch hydrodynamic damping, as can be seen in Figure [3.5c] and of surge pitch hydrodynamic damping, Figure [3.5a]. Regarding, heave hydrodynamic damping  $B_{33}$ , as can be seen in Figure [3.5b], the same conclusions as in  $A_{33}$  variation can be drawn, as  $B_{33}$  decreases as  $G$  obtains larger values for small values of wave frequencies, and the increase of  $G$  tends to have no effect on  $B_{33}$  as frequency values get higher.

The effect the porous parameter  $G$  has on the permeable body's heave motion and on its response is shown in Figure [3.7]. The mass of the body is equal to  $M = 3.62e^{-4}t$ , regardless of the  $G$  value, since the porous sidewall is assumed to be infinitesimally thin and have negligible mass. The cases of the fully impermeable cylinder and of the fully submerged plate are not taken under consideration as they attain completely different hydrostatic coefficients.

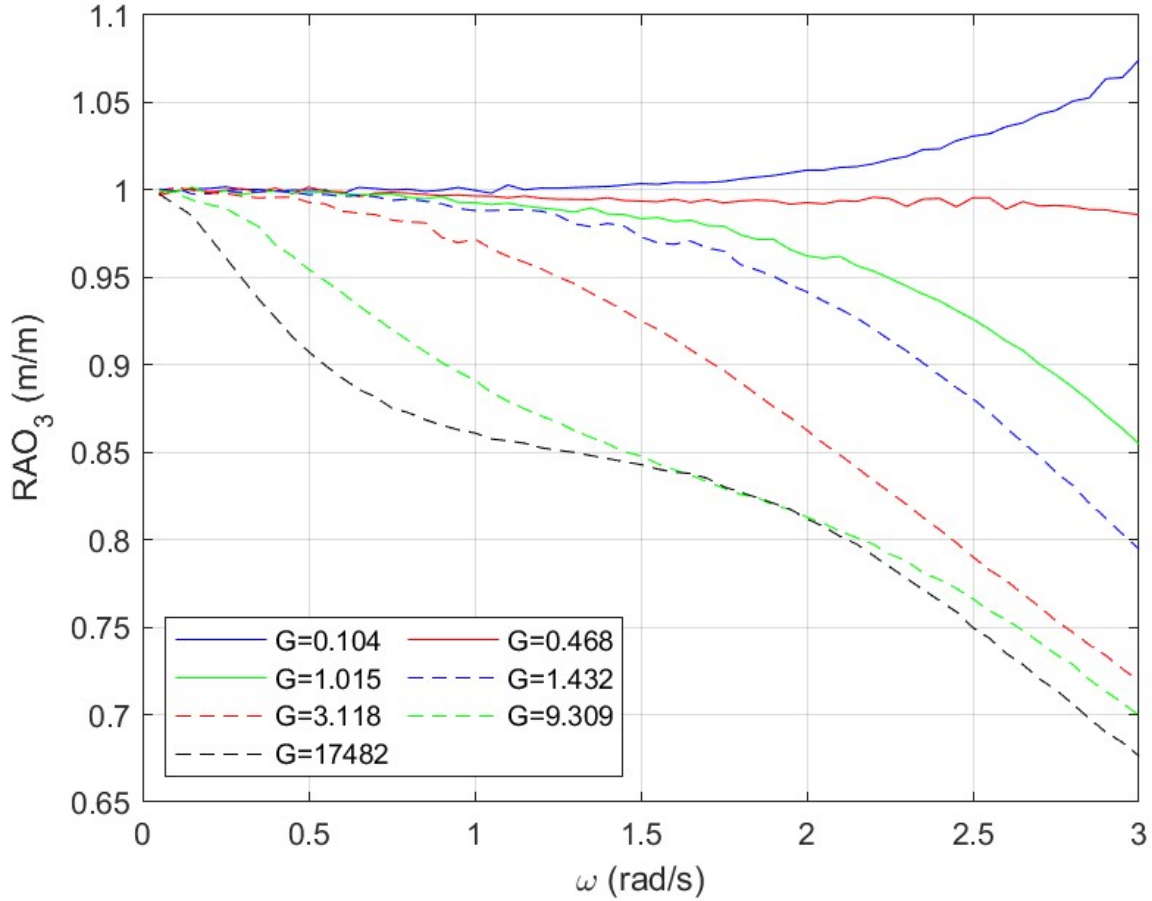


Figure 3.6

Figure 3.7: Motion  $RAO_3$  of the permeable cylindrical body against wave frequency  $\omega$  for various values of porous effect parameter  $G$

Regarding motion  $RAO_3$ , as can be seen in Figure [3.6] for  $\omega$  tending to zero  $RAO_3$  starts its variation from unity. However, as the  $G$  increases the permeable body's heave motion displacement decreases.

The following plots, Figure [3.8]-[3.11], show the effect of porous coefficient  $G$  on the hydrodynamic forces, moments and heave motion of the permeable cylindrical body, in various sea conditions. As can be seen in Figure [3.8], the surge hydrodynamic force,  $F_1$ , behaves reverse proportional to  $G$ . Furthermore, in every case of significant wave height  $H_s$ ,  $F_1$  decreases as peak wave period  $T_p$  obtains greater values and increases as significant wave height  $H_s$  does. Heave hydrodynamic force  $F_3$ , as seen in Figure [3.9], behaves reverse proportional to porous coefficient  $G$ . But, contrary to  $F_1$ ,  $F_3$  behaves proportional to peak wave period  $T_p$ , as well as to significant wave height  $H_s$ . Pitch hydrodynamic moment,  $F_5$  as depicted in Figure [3.10] follows the same trend as  $F_1$ .

In Figure [3.11], the effect of porous coefficient  $G$  on the permeable cylindrical body's heave motion  $RAO_3$  is depicted.  $RAO_3$  obtains smaller values as  $G$  increases, though its effect becomes less significant as peak wave period  $T_p$  increases. Additionally,  $RAO_3$  behaves proportional to both peak wave period  $T_p$  and significant wave height  $H_s$ .

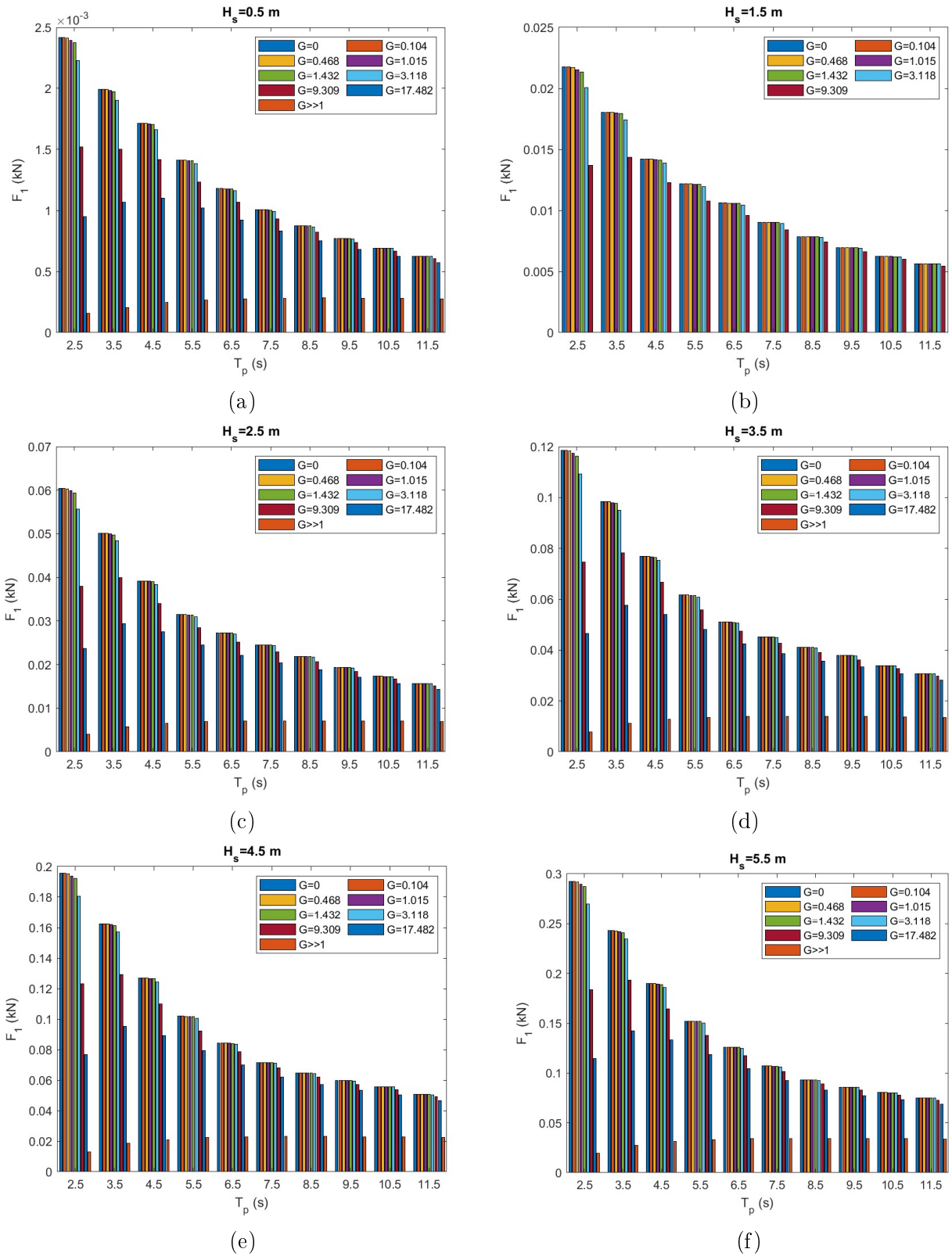


Figure 3.8: Values of  $F_1$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of porous effect parameter

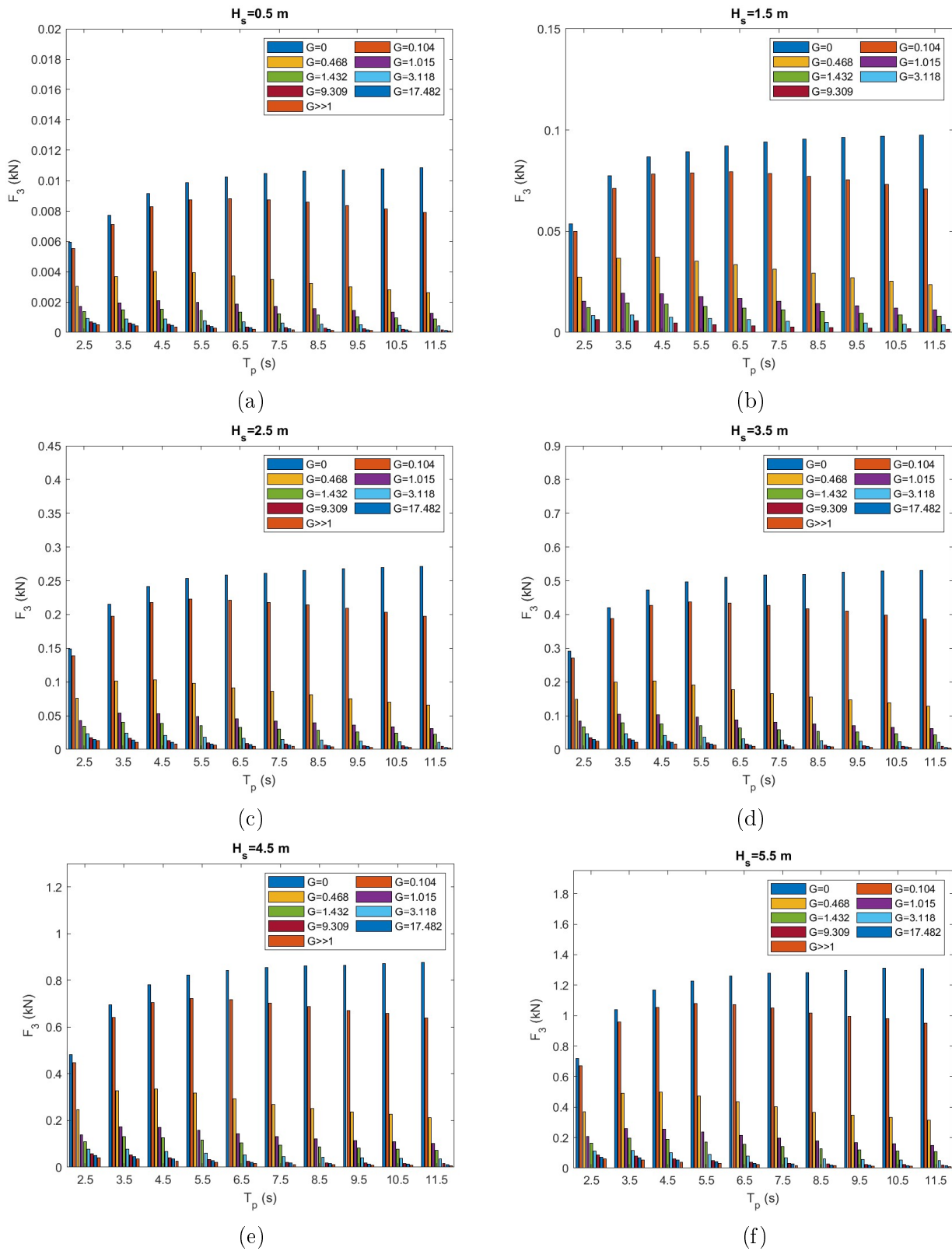


Figure 3.9: Values of  $F_3$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of porous effect parameter

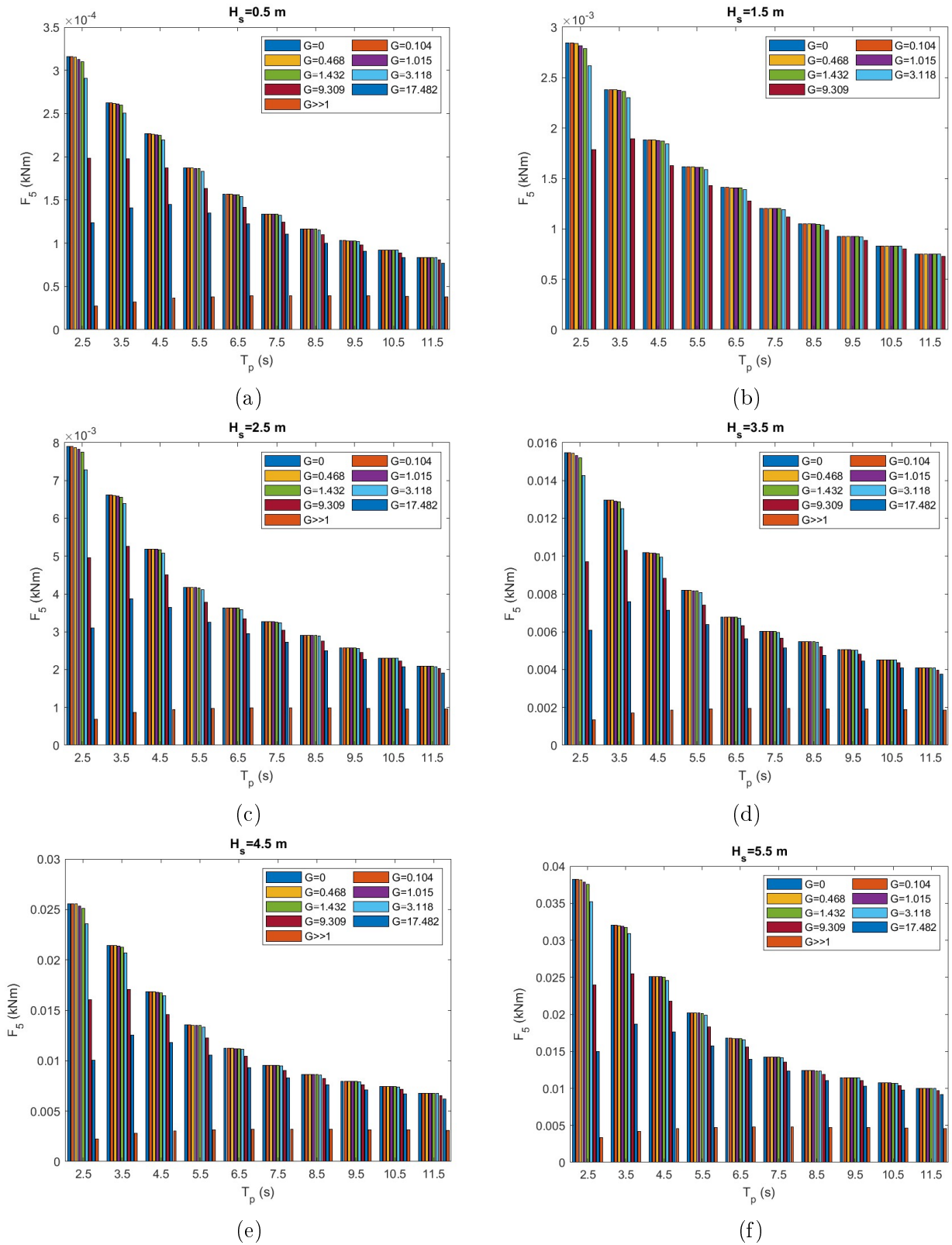


Figure 3.10: Values of  $F_5$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of porous effect parameter

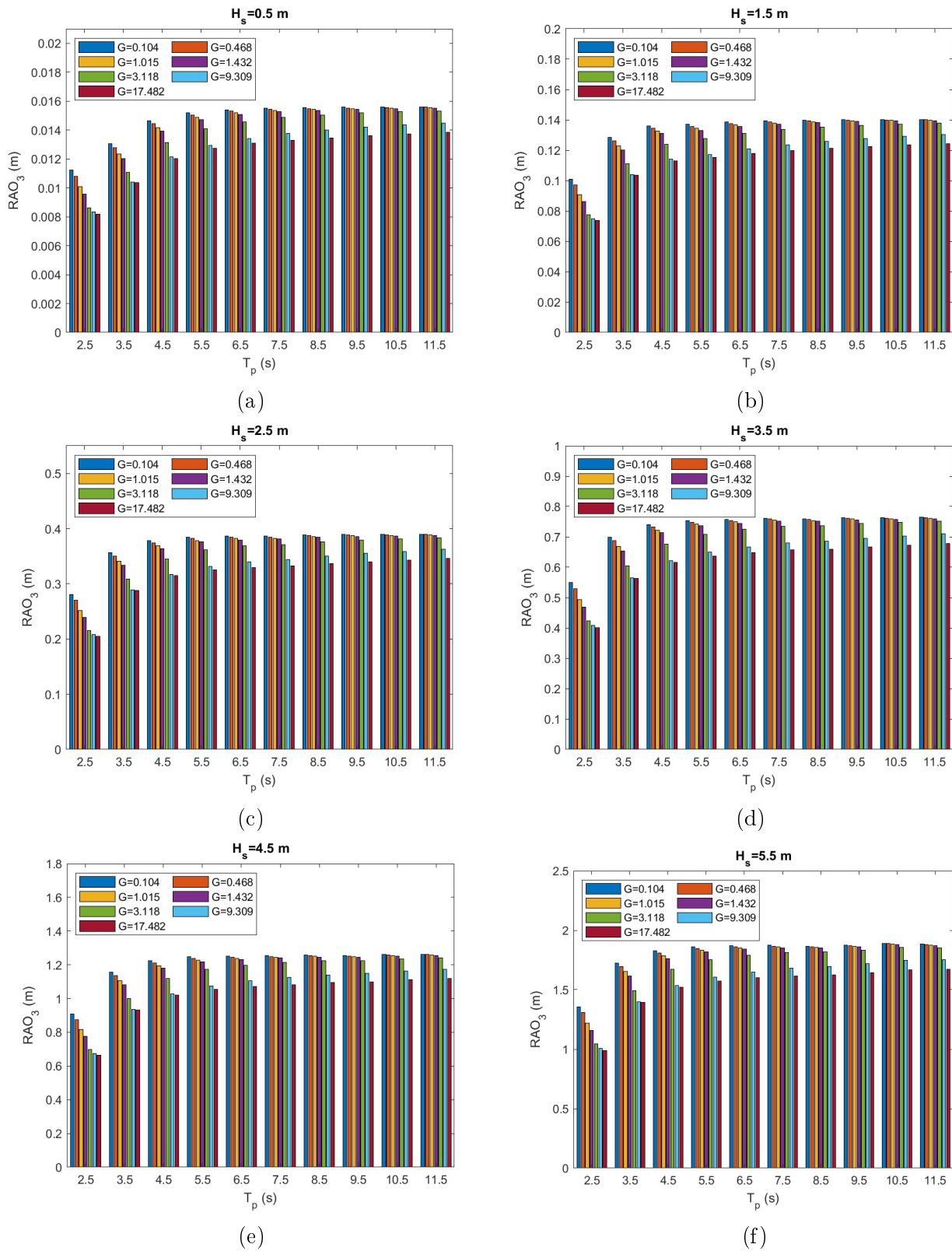


Figure 3.11: Values of  $RAO_3$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of porous effect parameter



### 3.2.2 Effect of the Radius

In this section, the effect the radius has on the hydrodynamic coefficients and exciting forces is examined. The body is, as aforementioned, a porous free floating cylinder of radius  $r$ , with an impermeable bottom of negligible thickness. It is subjected to waves propagating at zero angle in a constant water depth equal to  $d = 5$  m. The circular frequency of the waves is denoted by  $\omega$  with values ranging from  $\omega = 0.05$  rad/s to  $\omega = 3.0$  rad/s in steps of 0.05 rad/s. The characteristics, as in radius, mass, and mass moment of inertia relative to the free surface, of the examined test cases are seen below in 3. The center of gravity in all four cases is located in the vertical axis at 4.975 m below the free surface, and the porous coefficient  $G$  is real and equal to  $G = 1.432$ .

	$a$	$2a$	$3a$	$4a$
$r$ (m)	0.15	0.30	0.45	0.60
$M$ (t)	$3.62 * 10^{-4}$	0.0014	0.0033	0.0058
$I_z$ (tm <sup>2</sup> )	$4.07 * 10^{-6}$	$6.52 * 10^{-4}$	$3.30 * 10^{-4}$	0.001

Table 3: Characteristics of Test Cases

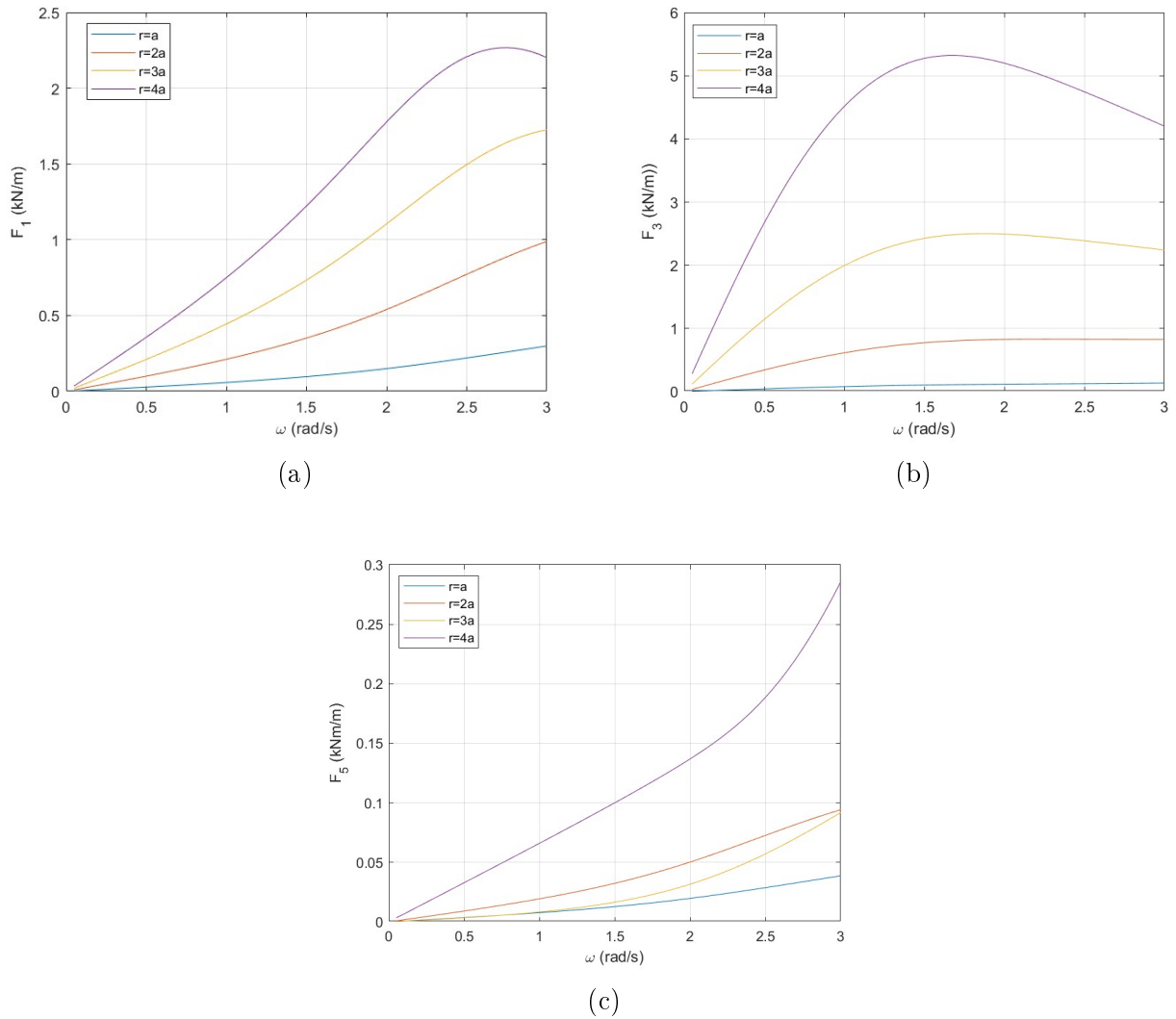


Figure 3.12: Exciting forces  $F_1$ ,  $F_3$  and moment  $F_5$  on the permeable cylindrical body against wave frequency  $\omega$  for various values of radius

The effect the value of the body's radius has on the exciting forces and overturning moment, is visualised in Figure [3.12]. Both surge  $F_1$  and heave  $F_3$  exciting forces, as seen in Figures [3.12a]-[3.12b], behave proportionally to radius values. It is also noted that in the case of heave,  $F_3$ , Figure [3.12b], the case of  $r = 4a$  the curve corresponding to the  $r = 4a$  case is characterised by a steeper slope and a rather larger local maximum in relation to the others. However, for the overturning moment,  $F_5$ , Figure [3.12c] the case is not the same. In the range of smaller frequencies  $\omega < 1$  the case of the  $r = 3a$  exhibits values close to  $r = a$  case and as the frequencies increase it tends to meet the  $r = 2a$  curve, while the  $r = 4a$  curve continues to exhibit similar behaviour as in surge and heave, see Figures [3.12a]-[3.12b], with the main difference being that it exhibits maximum values at higher frequencies.

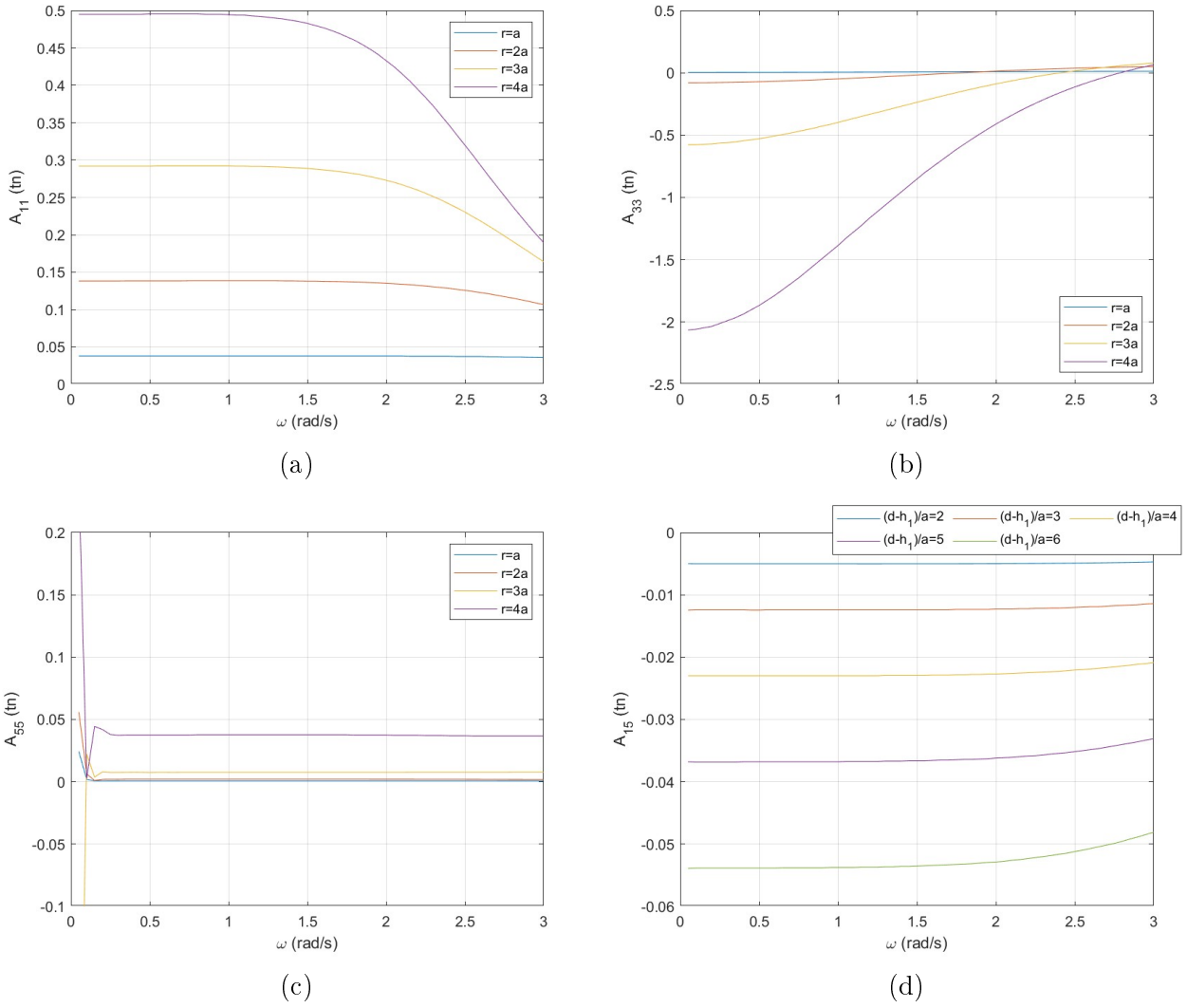


Figure 3.13: Hydrodynamic added mass  $A_{11}$   $A_{33}$ ,  $A_{55}$   $A_{15}$  of the permeable cylindrical body against wave frequency  $\omega$  for various values of radius.

Figure [3.13] depicts the hydrodynamic coefficient of added mass, of the examined permeable body, for various cases of radius value. Regarding the added mass in surge direction  $A_{11}$ , Figure [3.13a], in the frequency range of  $\omega < 1.5$  rad/s added mass is of proportionally greater value and constant for each case. As  $\omega$  increases  $A_{11}$  tends to decrease, more rapidly so, for higher radius values. In Figure [3.13b] the added mass in heave direction,  $A_{33}$  is depicted. It can be seen that,  $A_{33}$  behaves reverse proportionally with  $r$ , as it decreases as  $r$  increases. This effect is more prominent for  $\omega < 2.5$  rad/s as for frequencies higher than that,  $A_{33}$  tends to zero in all the examined cases, making the effect of  $r$  negligible. The added mass in pitch direction  $A_{55}$ , is depicted on Figure [3.13c]. It can be noted, that the curves of  $r = a$ ,  $r = 2a$ ,  $r = 4a$  attain a local maxima, that is of higher value for greater  $r$  values in the lower frequency range  $\omega < 0.2$  rad/s whereas  $r = 3a$  attains a local minima in the same range. For greater values of  $\omega$ ,  $A_{55}$  behaves proportionally with  $r$ . In case of surge pitch added mass  $A_{15}$ , Figure [3.13d], the curve corresponding to  $r = 4a$  follows a similar behaviour as the one in surge, see Figure [3.13a], while it also holds true that for greater values of  $r$ ,  $A_{15}$  increases as well, except for the case of  $r = 2a$  which for the whole frequency range has a negative value of  $A_{15}$ . It also holds

true that  $A_{15}=A_{51}$ .

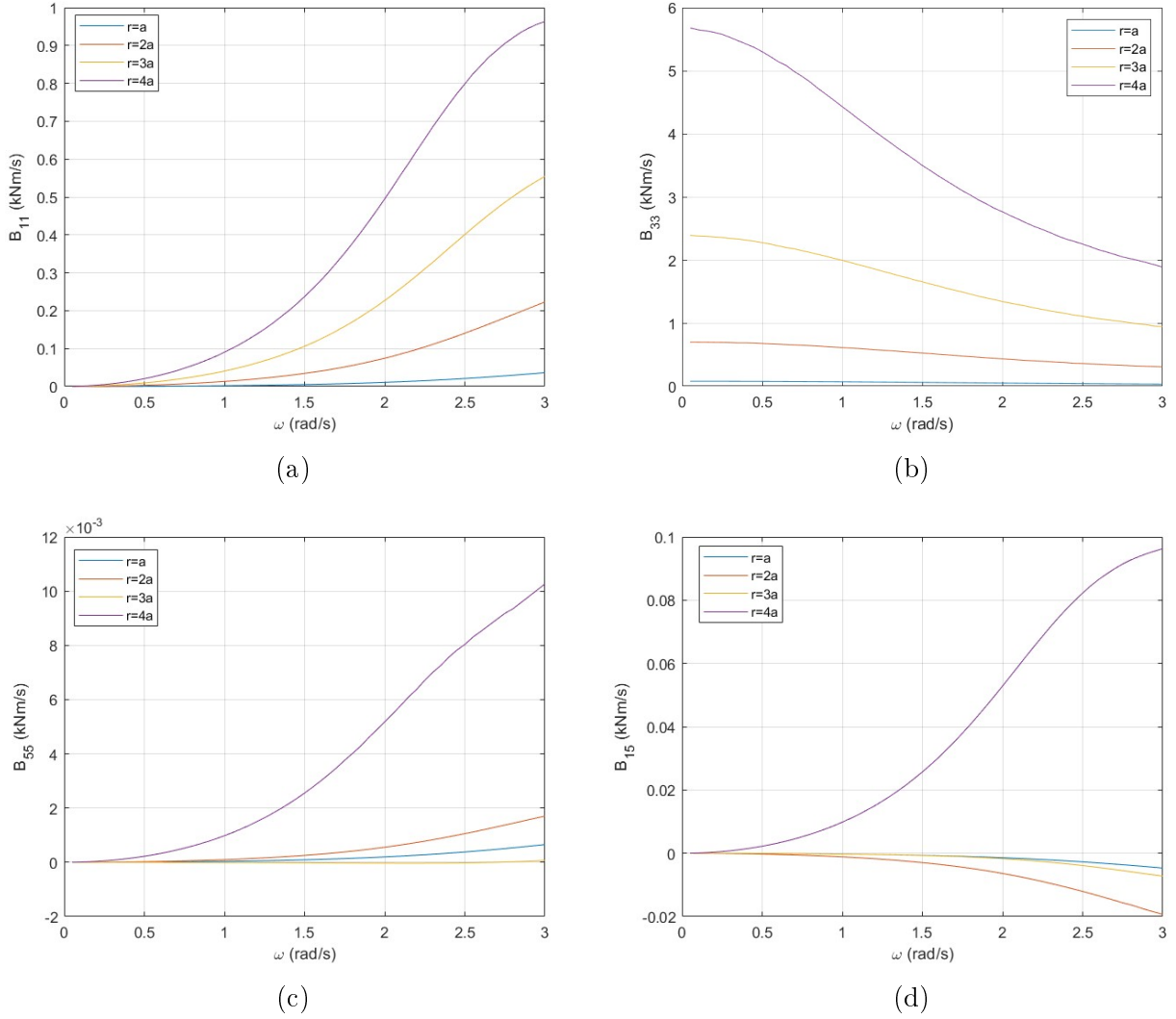


Figure 3.14: Hydrodynamic damping  $B_{11}$   $B_{33}$ ,  $B_{55}$   $B_{15}$  of the permeable cylindrical body against wave frequency  $\omega$  for various values of radius.

In Figure [3.14] the effect the value of the permeable cylindrical body's radius has on the hydrodynamic damping coefficient, against frequency, is depicted. The hydrodynamic damping coefficient in the surge direction  $B_{11}$ , Figure [3.14a], behaves proportionally with  $r$ , as  $r$  increases so does  $B_{11}$ , and this effect is more prominent as  $\omega$  values get higher. In the range of  $\omega < 0.3$  rad/s the effect of  $r$  is practically negligible. As can be seen in Figure [3.14b], the hydrodynamic damping in the heave direction  $B_{33}$ , behaves proportionally to  $r$  similarly with the case of  $B_{11}$ , see Figure [3.14a], with the key difference being, that in the case of  $B_{33}$  the effect of  $r$  is prominent in the whole frequency range. In the case of damping on the pitch direction  $B_{55}$ , Figure [3.14c], however, the aforementioned pattern is not followed. Here, the lowest values of  $B_{55}$  correspond to the case of  $r = 3a$  presenting values near zero for frequencies smaller than  $\omega < 1$  rad/s and attains negative values for the rest of the frequency range. Similarly, in the case of surge pitch damping  $B_{15}$  the case of  $r = 4a$  exhibits a similar behaviour as in surge  $B_{11}$ , see Figure [3.14a], while the rest behave reverse proportionally and as  $r$  increases  $B_{15}$  attains

smaller values which for  $\omega > 07 \text{ rad/s}$   $B_{15}$  are negative.

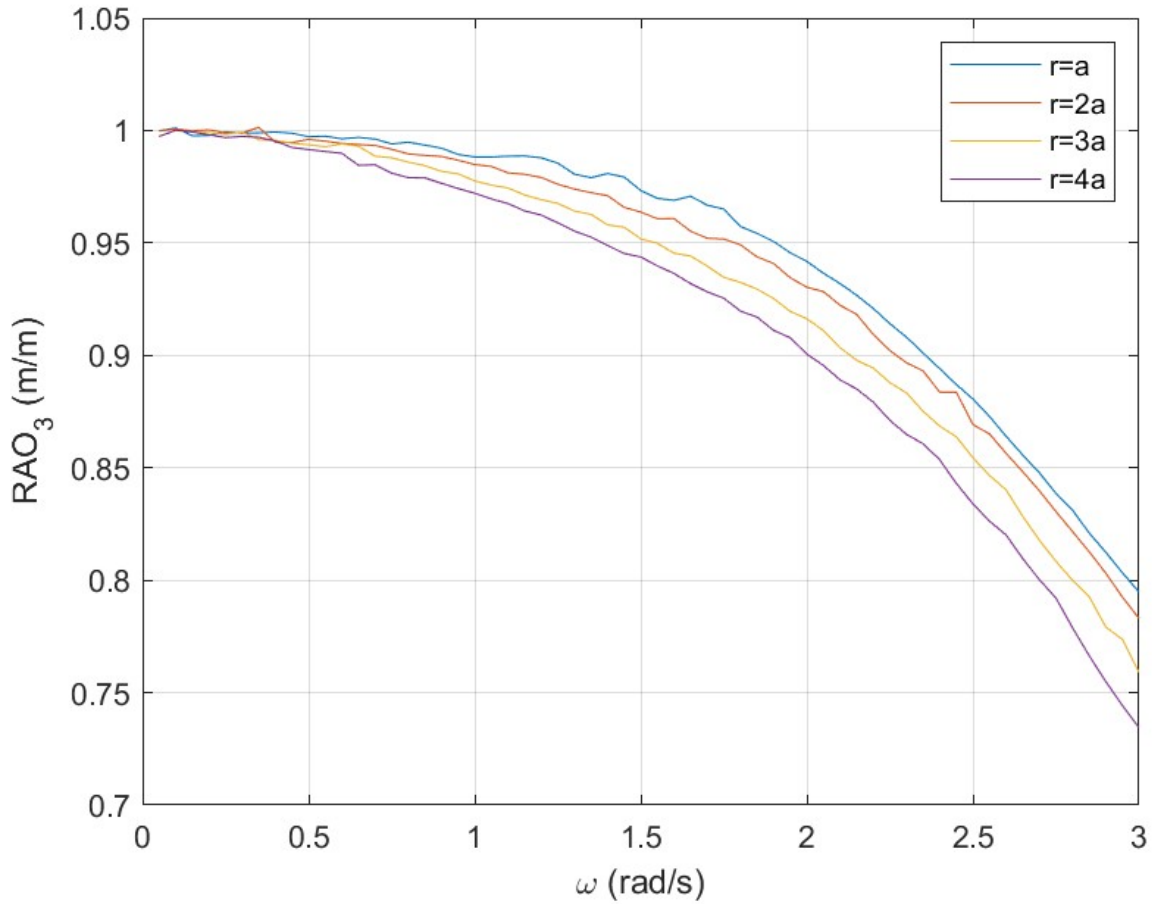
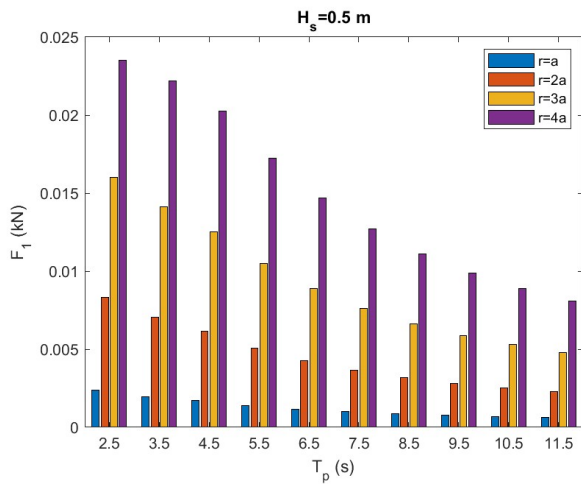


Figure 3.15

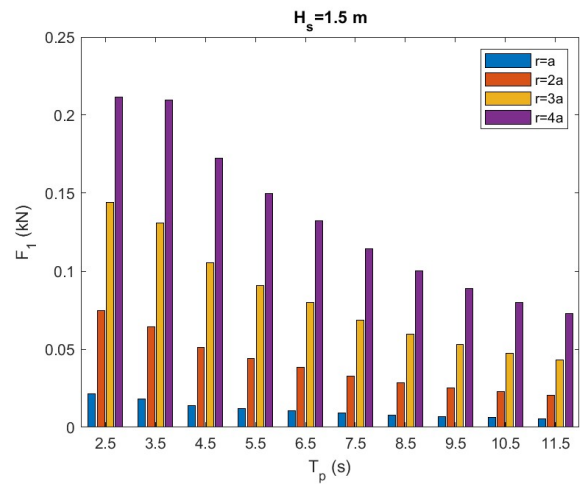
Figure 3.16: Motion  $RAO_3$  of the permeable cylindrical body against wave frequency  $\omega$  against wave frequency  $\omega$  for various values of radius

As it is shown in Figure [3.15] the heave motion  $RAO$  for  $\omega$  tending to zero, starts its variation from unity. As  $\omega$  increases, it is evident that for radius of greater values the heave motion displacement decreases.

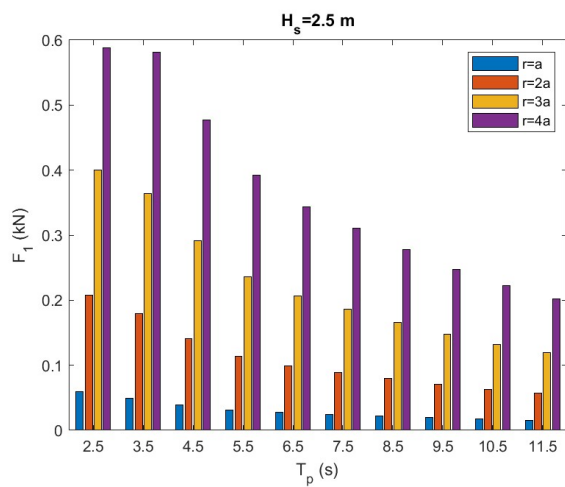
The following plots, Figure [3.17]-[3.20], show the effect of radius on the hydrodynamic forces, moments and heave motion of the permeable cylindrical body, in various sea conditions. As can be seen in Figure [3.17], the surge hydrodynamic force,  $F_1$ , behaves proportional to radius. Furthermore, in every case of significant wave height  $H_s$ ,  $F_1$  increases until peak wave period obtains a value of  $T_p = 2.5 \text{ s}$  and then gradually decreases.



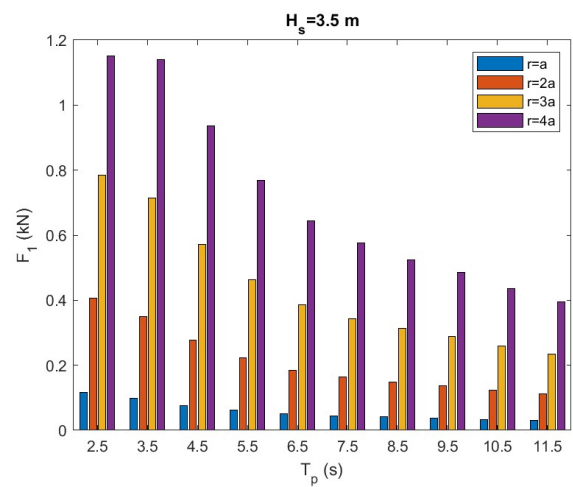
(a)



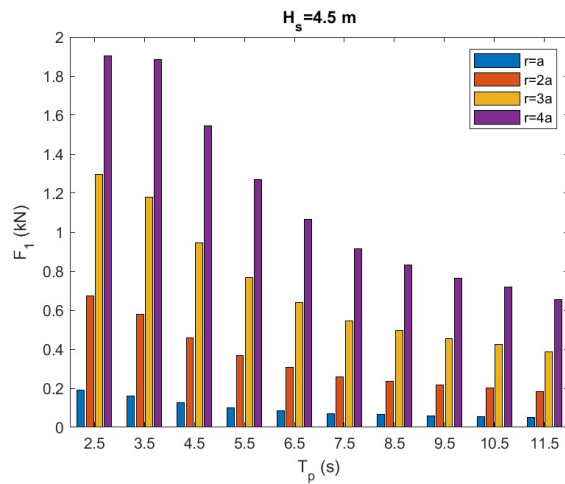
(b)



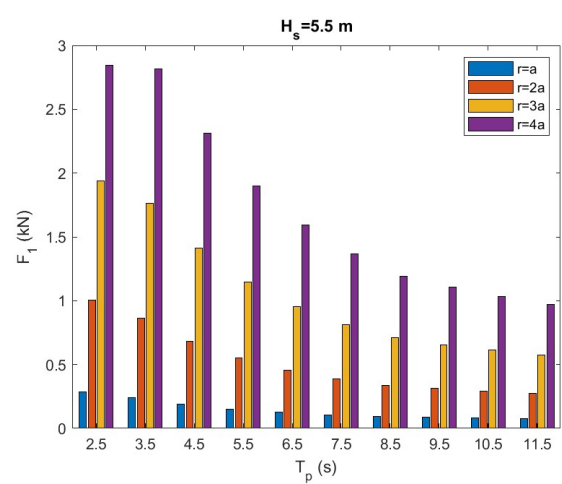
(c)



(d)

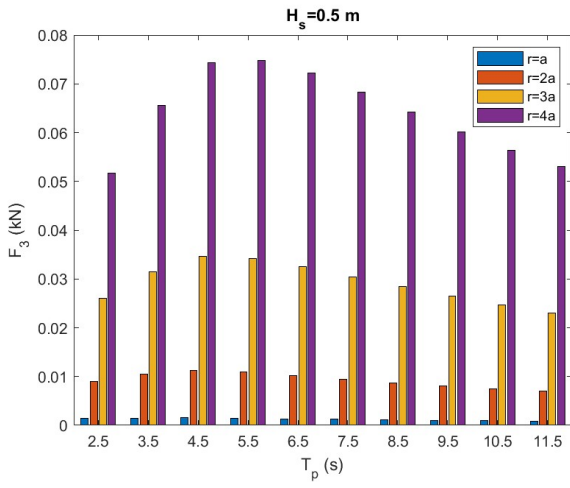


(e)

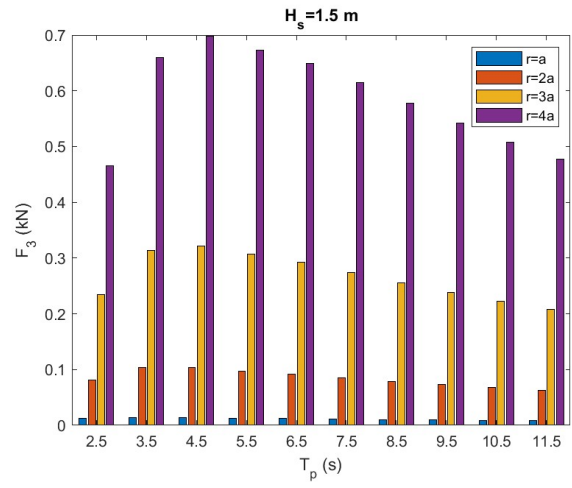


(f)

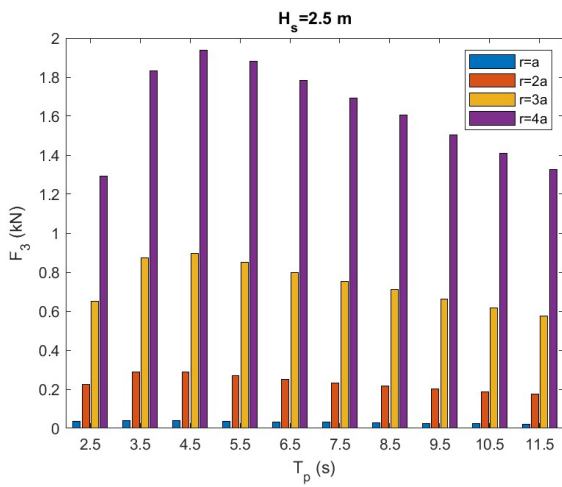
Figure 3.17: Values of  $F_1$  against peak wave period  $T_p$  and significant wave height  $H_s$  for different values of radius



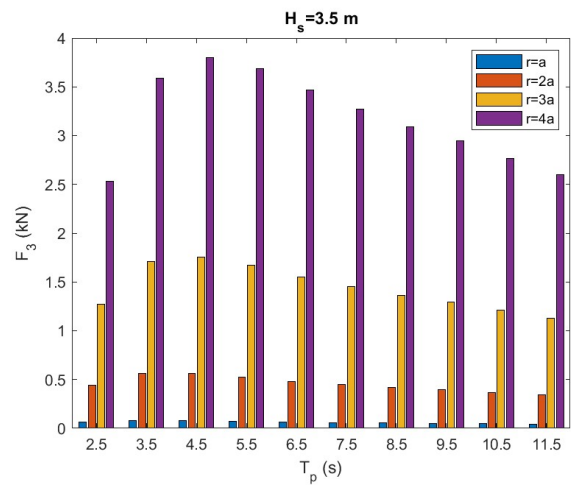
(a)



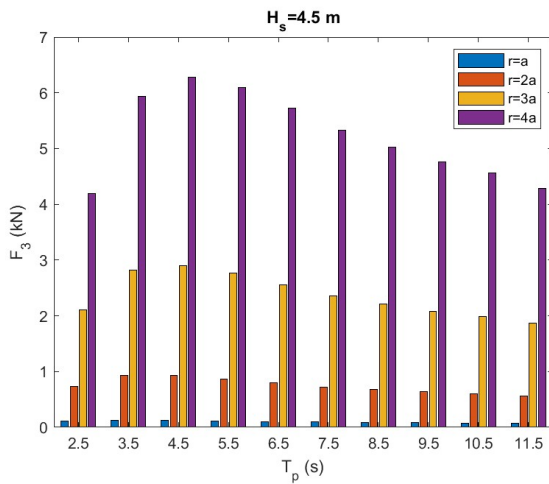
(b)



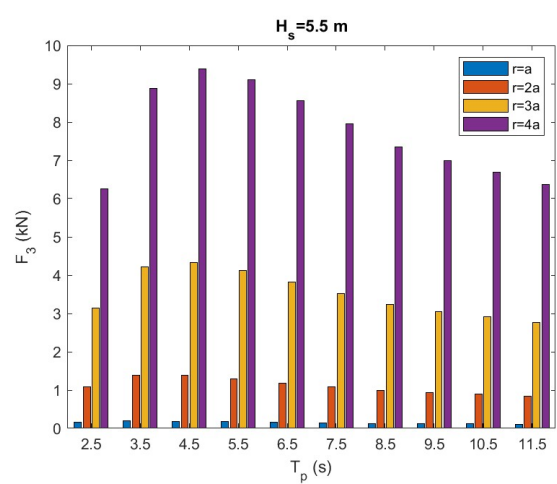
(c)



(d)



(e)



(f)

Figure 3.18: Values of  $F_3$  against peak wave period  $T_p$  and significant wave height  $H_s$  for different values of radius

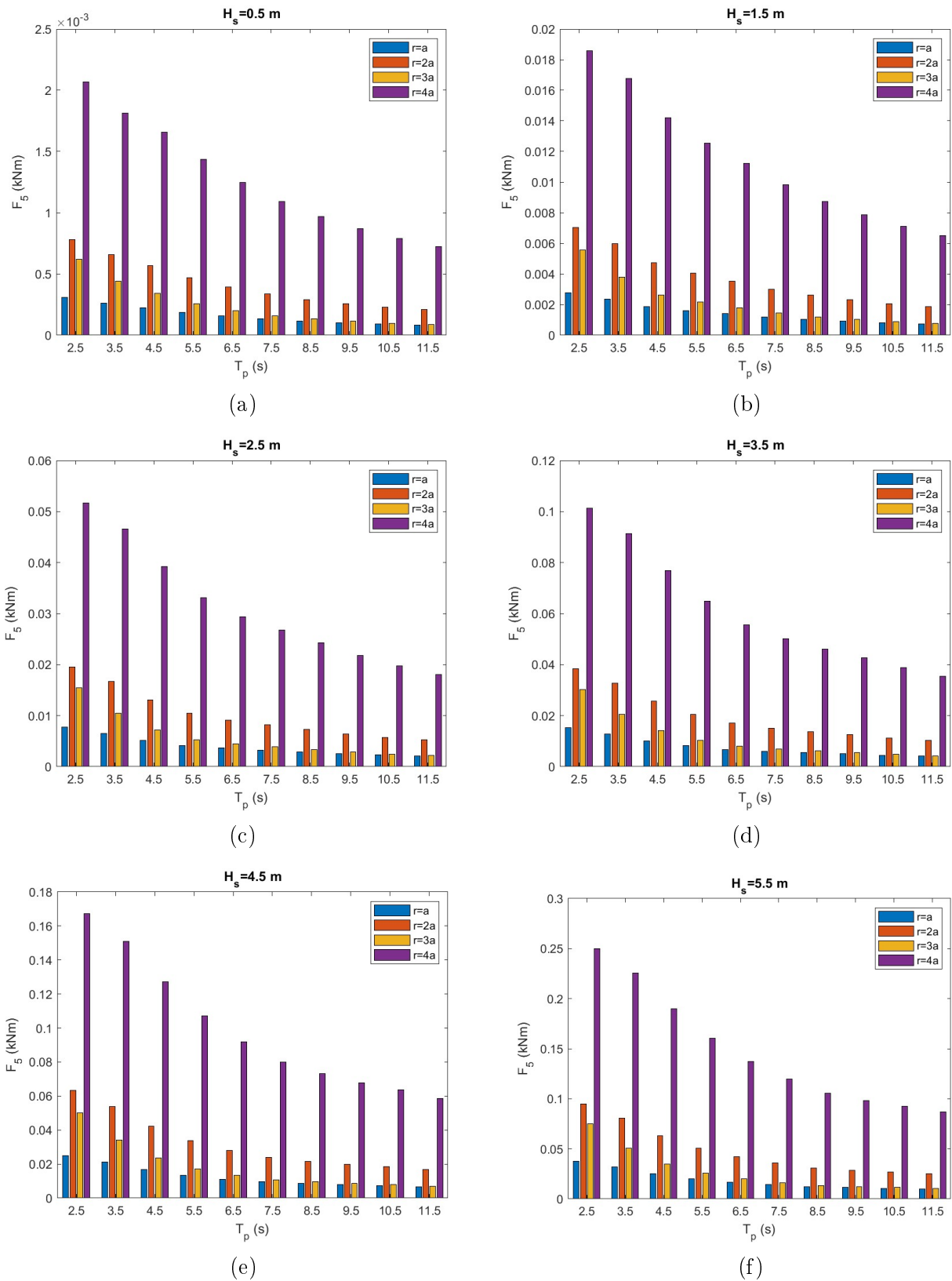


Figure 3.19: Values of  $F_5$  against peak wave period  $T_p$  and significant wave height  $H_s$  for different values of radius



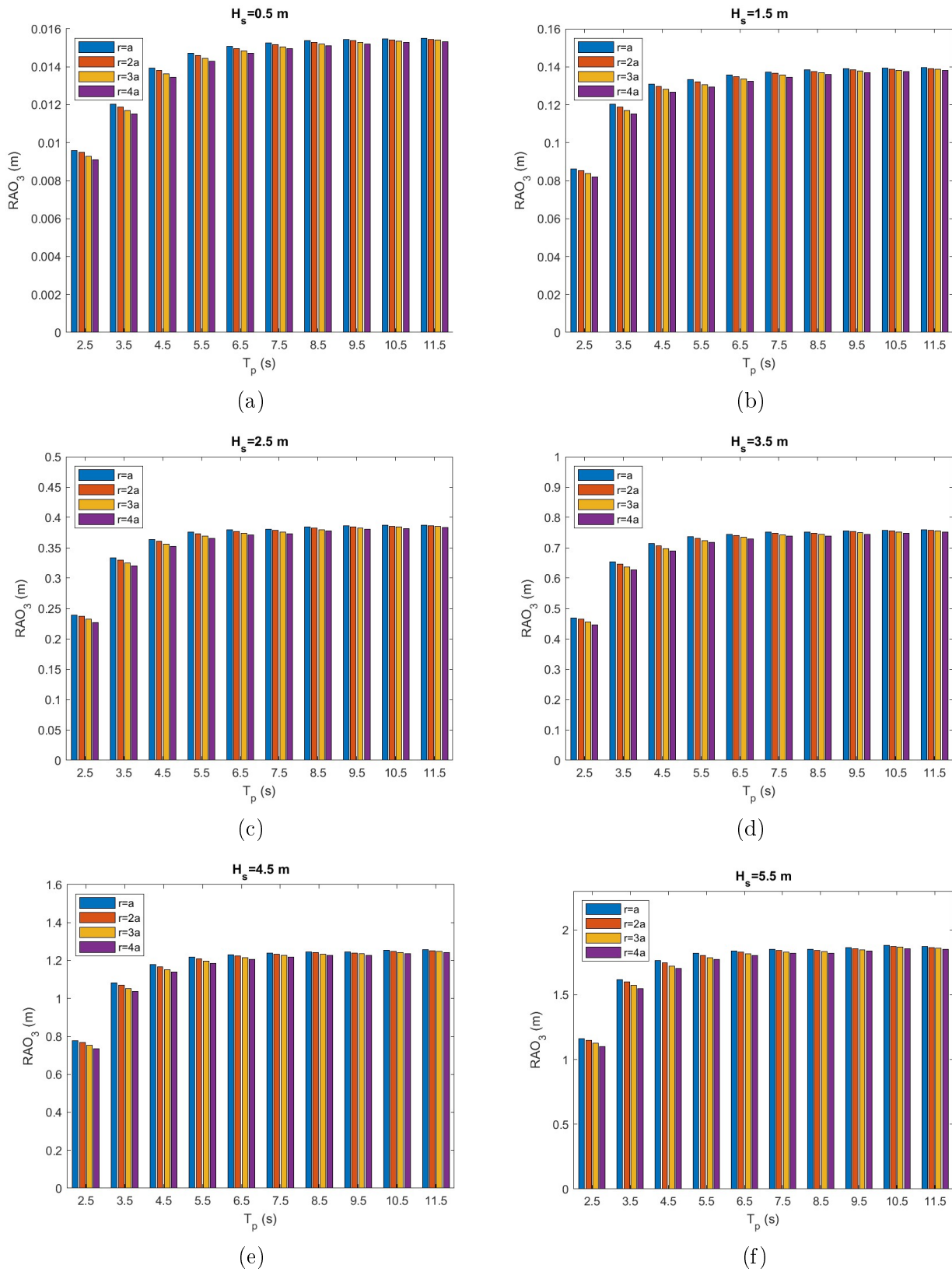


Figure 3.20: Values of  $RAO_3$  against peak wave period  $T_p$  and significant wave height  $H_s$  for different values of radius

Heave hydrodynamic force  $F_3$  as depicted in Figure [3.18], is also proportional to radius. In

most case of significant wave height,  $H_s$ ,  $F_3$  increases until peak wave period obtains a value of  $T_p = 4.5$  s and then gradually decreases. In the case of  $H_s = 0.5$  m the maximum value of  $F_3$  corresponds to period of value  $T_p = 5.5$  s. As can be observed, in Figure [3.19],  $F_5$  does not behave proportional to radius. In this case and in accordance to Figure [3.12], in case of  $r = 2a$  hydrodynamic moment  $F_5$  obtains greater values for every  $T_p$ , compared to  $r = 3a$ . Additionally, greater values  $F_5$  correspond to lower values of peak wave period  $T_p$ .

In Figure [3.15], the effect of radius on the heave motion  $RAO_3$  is shown.  $RAO_3$  behaves reverse proportionally to radius, and as peak wave period obtains greater values the effect of radius becomes negligible.

### 3.2.3 Effect of Draught

In this section, the effect the draught has on the hydrodynamic coefficients and exciting forces is examined. The body is, as aforementioned, a porous free floating cylinder of radius  $r$ , with an impermeable bottom of negligible thickness. It is subjected to waves propagating at zero angle in a constant water depth equal to  $d = 5$  m. The circular frequency of the waves is denoted by  $\omega$  with values ranging from  $\omega = 0.05$  rad/s to  $\omega = 3.0$  rad/s in steps of 0.05 rad/s. The porous coefficient is real and equal to  $G = 1.432$ . The examined cases and the locations of corresponding center of gravity (CoG) at the vertical axis  $Z_G$ , of each case is shown below in Table (4).

$(d - h_1)/a$	2	3	4	5	6
$Z_G$ (m)	4.697	4.547	4.397	4.247	4.097

Table 4: Examined cases of draught effect and location of center of gravity  $Z_G$  at the vertical axis

The effect the value of draught has on the body's exciting forces and moment is depicted in Figure [3.21]. Both surge  $F_1$  exciting force and pitch  $F_5$  moment, as seen in Figures [3.21a]-[3.21c] behave proportionally to the value of draught. As the draught increases so do the exciting force and moment. However, this is not the case for the heave exciting force  $F_3$ . In this case, as seen in Figure [3.21b],  $F_3$  behaves reverse proportionally to the draught values. The effect of draught for the cases of  $(d - h_1)/a = 4, 5, 6$  is more prominent in the frequency ranges of  $\omega < 1.5$  rad/s and  $\omega > 2$  rad/s.

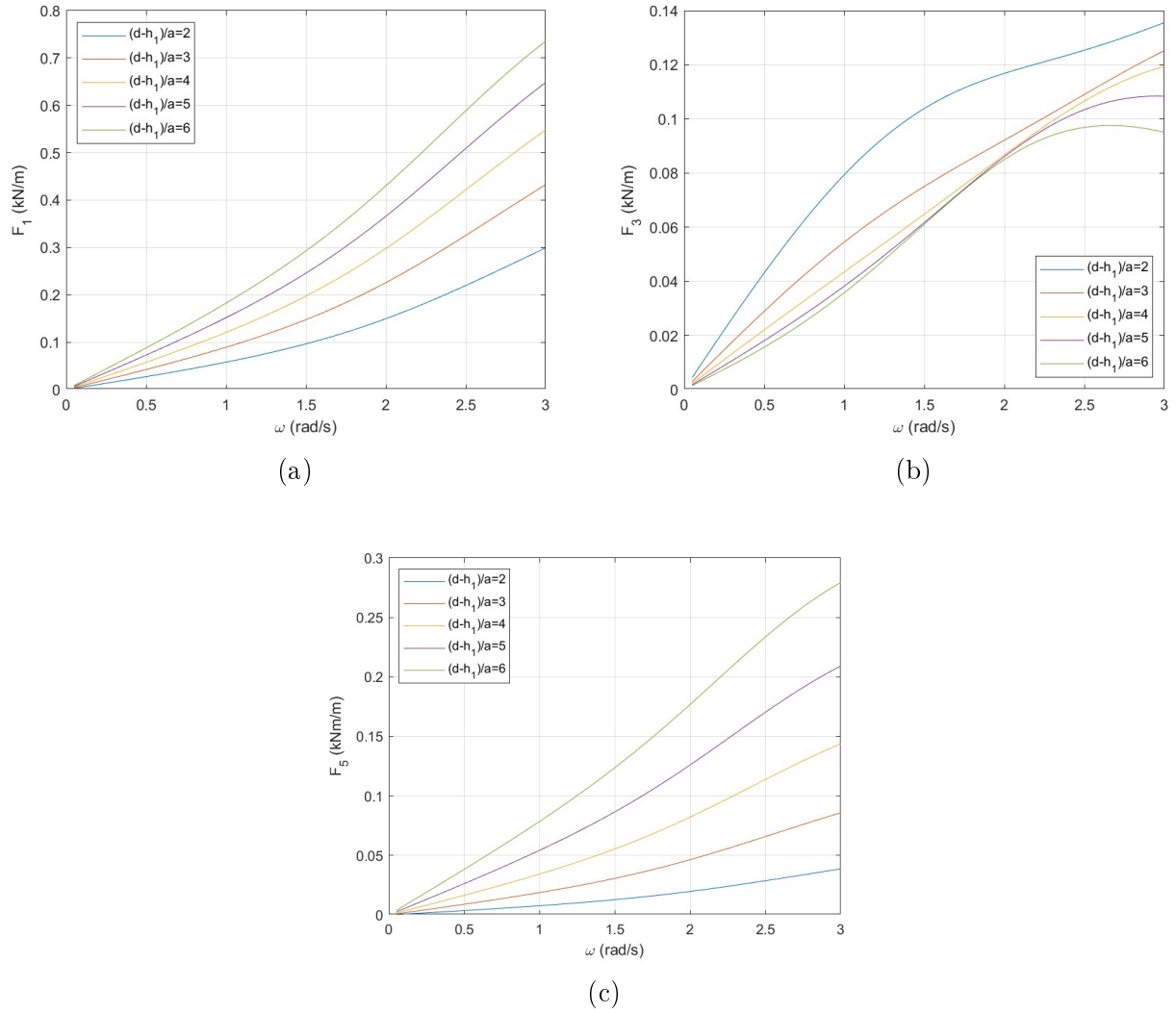


Figure 3.21: Exciting forces  $F_1$ ,  $F_3$  and moment  $F_5$  on the permeable cylindrical body against wave frequency  $\omega$  for various values of draught

The effect the draught has on the hydrodynamic added mass of the permeable body is depicted in Figure [3.22]. In the case of surge added mass  $A_{11}$  and surge pitch added mass  $A_{15}$ , Figure [3.22a] and Figure [3.22d] respectively, is clear that as the draught increases so does the added mass. The same holds true for the case of heave added mass  $A_{33}$  but in this case, as  $\omega$  increases, the curves of  $A_{33}$  for different draught values tend to collide. In case of the pitch added mass  $A_{55}$ , Figure [3.22c] is worth noted that while in the cases of  $(d-h_1)/a = 2, 4, 6$  the corresponding added mass curves obtain local minima in the range of  $\omega \simeq 0.1\text{rad/s}$  the remaining cases of  $(d-h_1)/a = 3, 5$  obtain local maxima in the same frequency region.

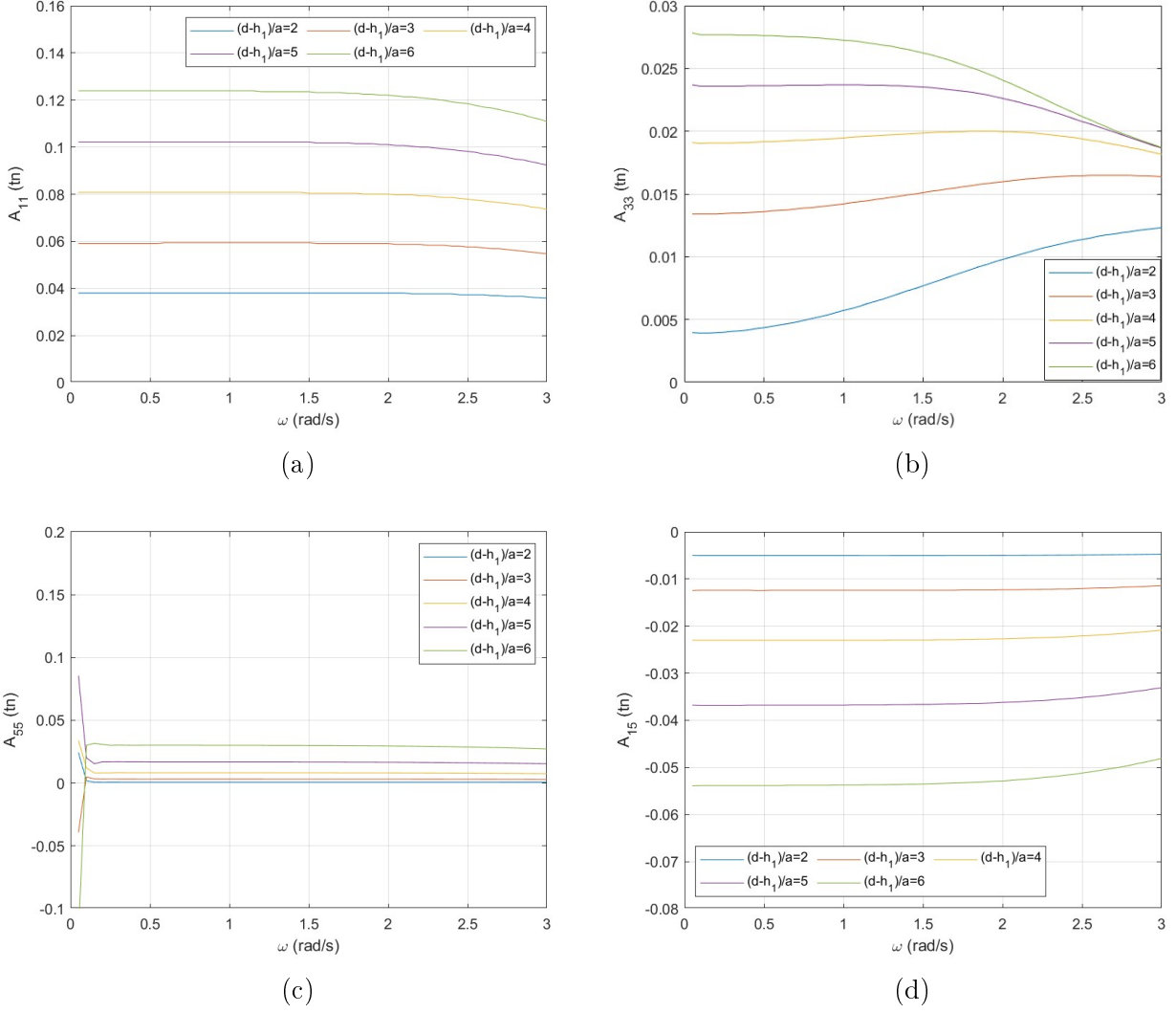


Figure 3.22: Hydrodynamic added mass  $A_{11}$   $A_{33}$ ,  $A_{55}$   $A_{15}$  of the permeable cylindrical body against wave frequency  $\omega$  for various values of draught.

In Figure [3.23] the effect the draught has, on the permeable cylindrical body's hydrodynamic damping coefficients, is depicted. In the cases of surge,  $B_{11}$ , pitch,  $B_{55}$  and surge pitch  $B_{15}$ , Figures [3.23a],[3.23c] and [3.23d] respectively, the hydrodynamic damping coefficients exhibit the same behaviour. In the lower frequency range,  $\omega < 0.5 \text{ rad/s}$  the value of draught has no significant impact on hydrodynamic damping, and as  $\omega$  increases, the hydrodynamic damping coefficients behave proportionally to the draught, as the draught increases so does the hydrodynamic damping. This, however, is not the case for the heave hydrodynamic damping coefficient  $B_{33}$  as shown in Figure [3.23b]. In this case, for frequencies smaller than  $\omega < 1.5 \text{ rad/s}$   $B_{33}$  behaves reverse proportionally to draught, and for  $\omega > 1.5 \text{ rad/s}$  the curve that corresponds to  $(d-h_1)/a = 6$  follows an increasing trend whereas the rest of the cases exhibit a steady decline.

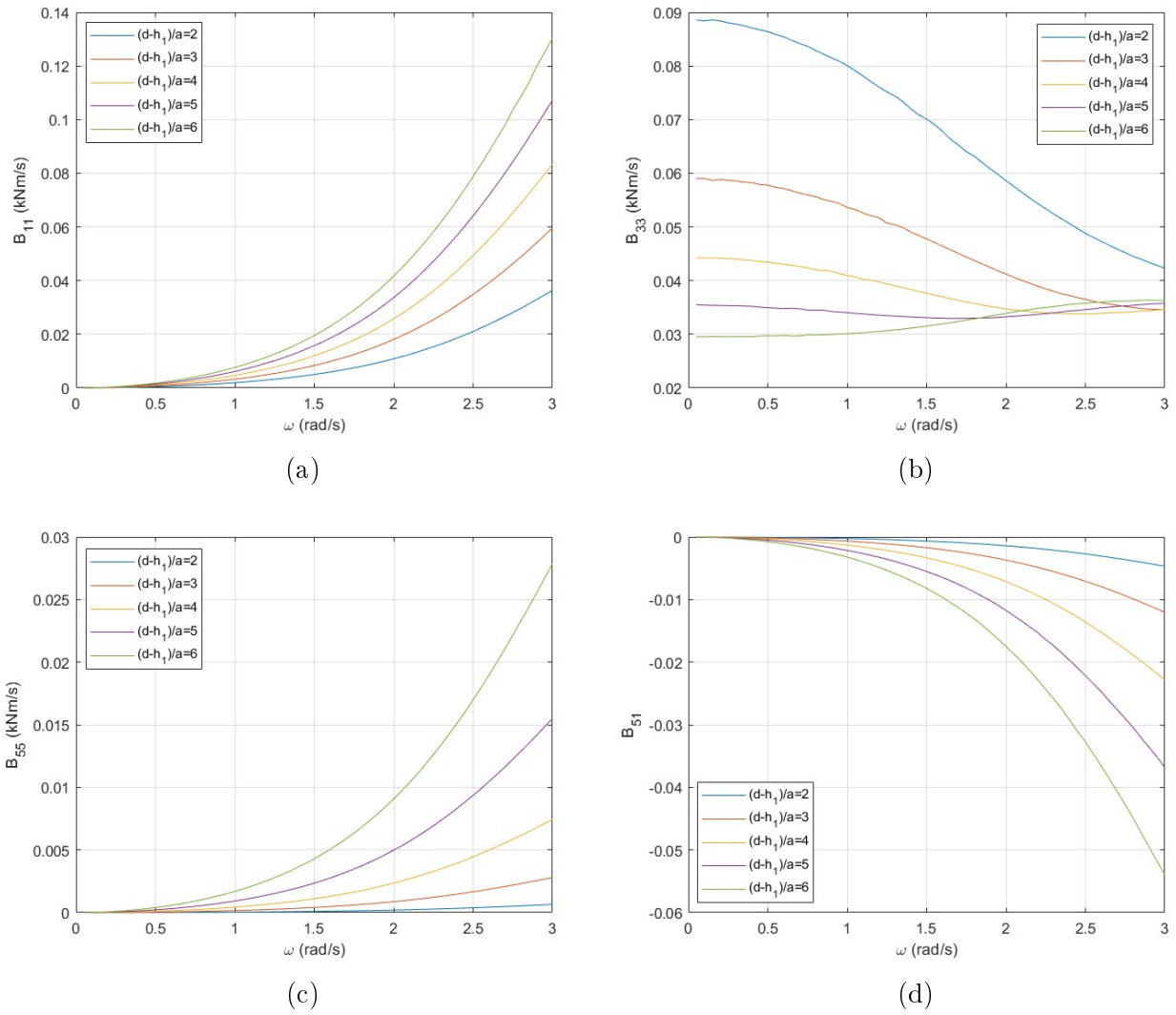


Figure 3.23: Hydrodynamic damping  $B_{11}$   $B_{33}$ ,  $B_{55}$   $B_{15}$  of the permeable cylindrical body against wave frequency  $\omega$  for various values draught.

In Figure [3.24], the effect different values of draught have on the permeable cylindrical body's heave motion  $RAO$ , is depicted. For  $\omega$  tending zero,  $RAO_3$  obtains a value of 1 for all the examined cases. As the  $\omega$  increases, it is clear that for greater values of draught the heave motion displacement decreases.

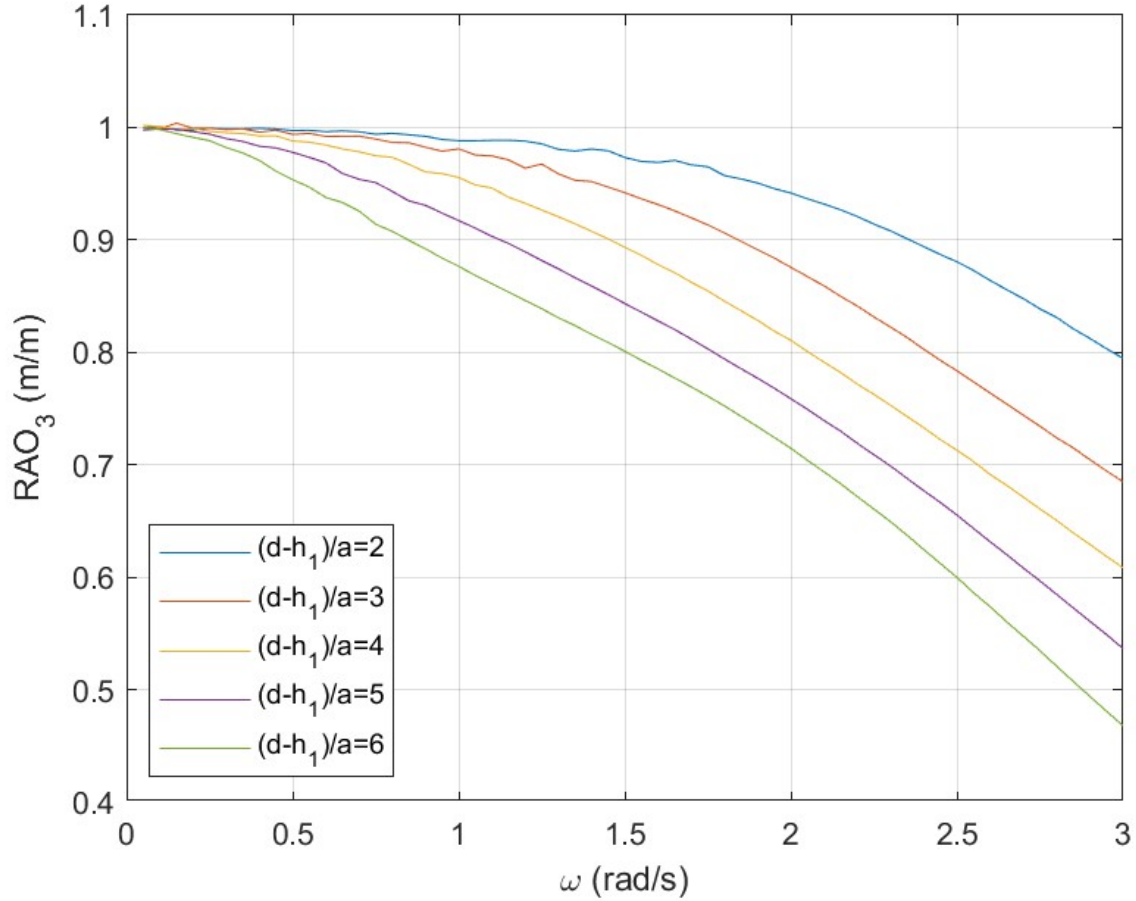


Figure 3.24

Figure 3.25: Motion  $RAO_3$  of the permeable cylindrical body against wave frequency  $\omega$  for various values of draught

The following plots, Figure [3.26]-[3.29], show the effect of draught on the hydrodynamic forces, moments and heave motion of the permeable cylindrical body, in various sea conditions. Regarding the hydrodynamic forces  $F_1$  and  $F_5$  Figures [3.26] and [3.28], it is evident that they behave proportional to the draught. It is also worth noted that the force and moment attain larger values as the significant wave height increases. Contrary, as the peak wave period increases,  $F_1$  and  $F_5$  decrease. In Figure [3.27] the effect of draught on heave force is depicted. In this case, the force behaves reverse proportional to the body, meaning that for higher values of draught,  $F_3$  decreases. The peak values of heave  $F_3$  in most sea states, are obtained for values of peak wave period  $T_p \simeq 3.5$  s. The exception being the case of  $H_s 0.5$  m, Figure [3.29] where  $F_3$  obtains peak values for peak wave period of  $T_p = 4.5$  s.

Regarding heave motion  $RAO_3$ , as it is shown in Figure [3.29], it also behaves proportional to draught. The higher values of  $RAO_3$  are obtained for the extreme weather conditions of  $H_s = 5.5$  m and  $T_p = 11.5$  s. It is also worth noted, that draught seems to have less effect on  $RAO_3$  as  $T_p$  increases.

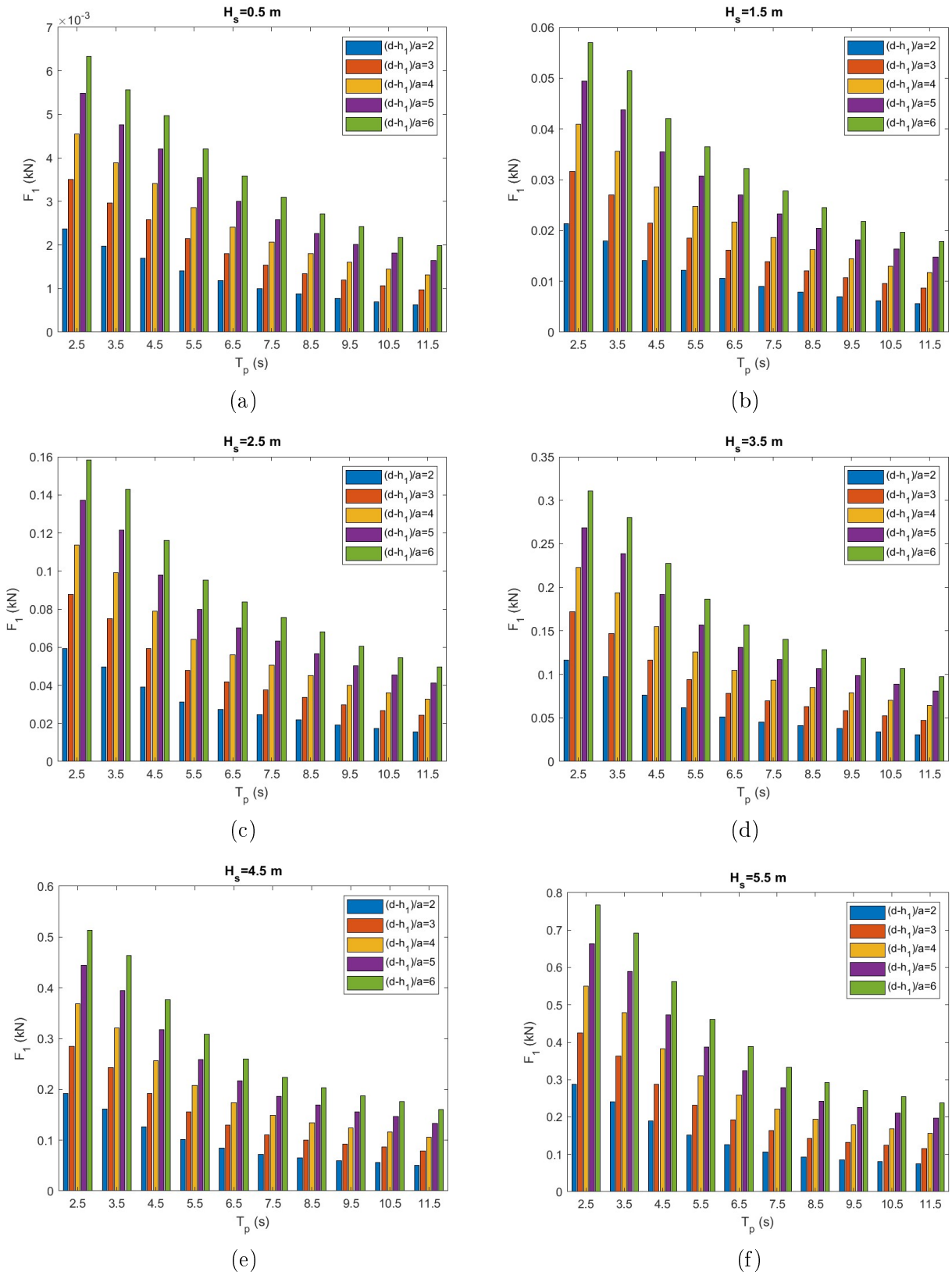


Figure 3.26: Values of  $F_1$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of draught

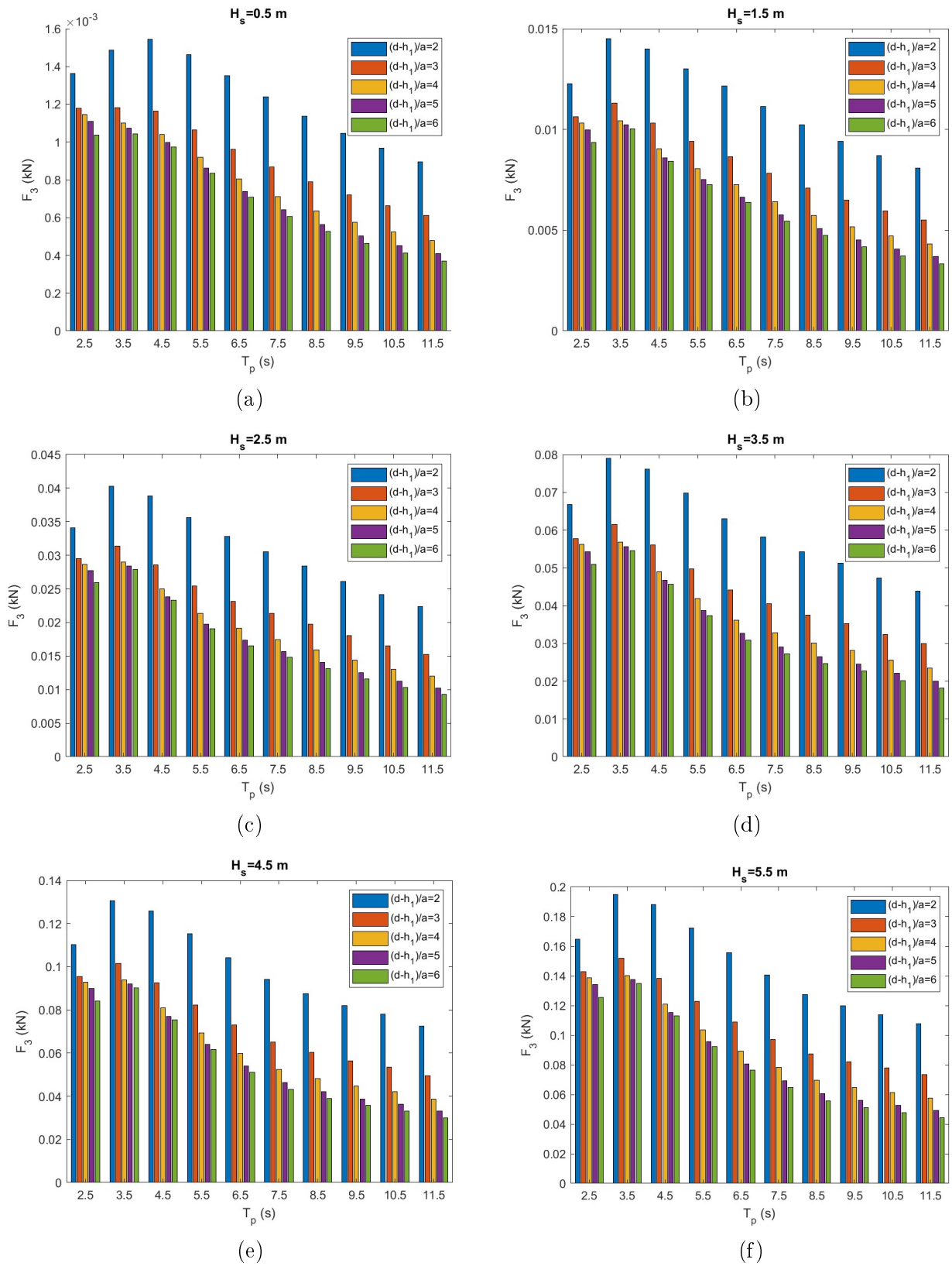
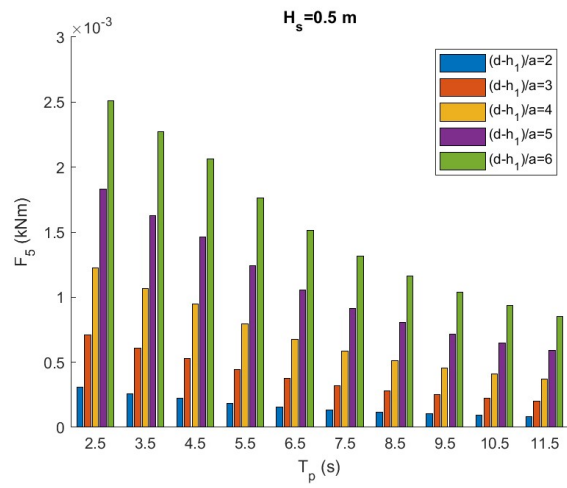
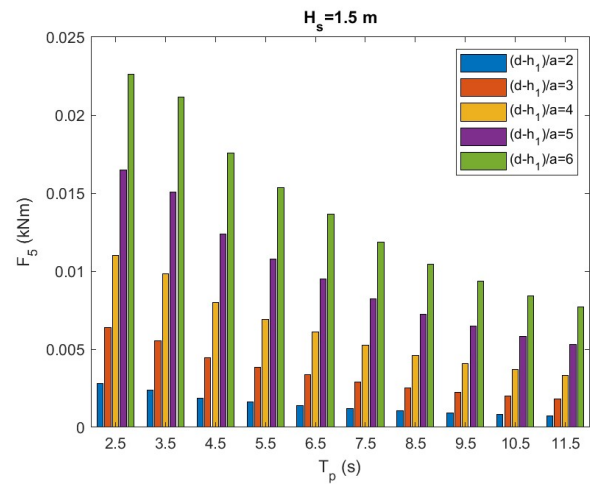


Figure 3.27: Values of  $F_3$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of draught

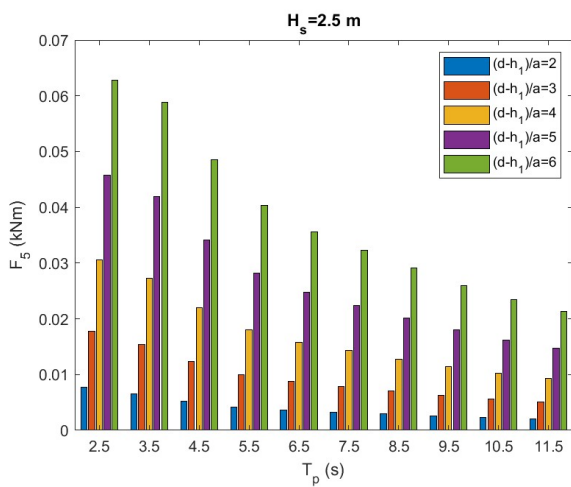




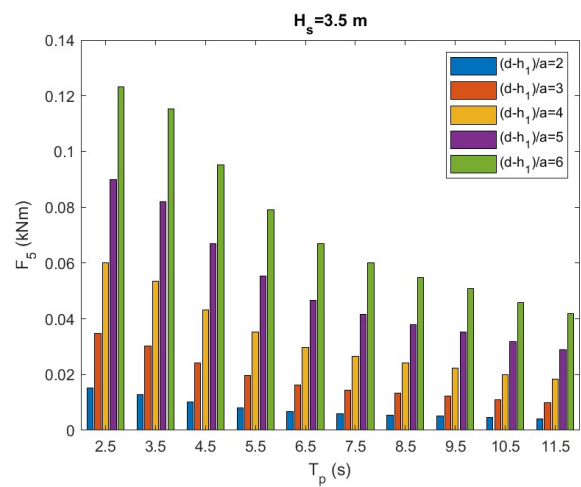
(a)



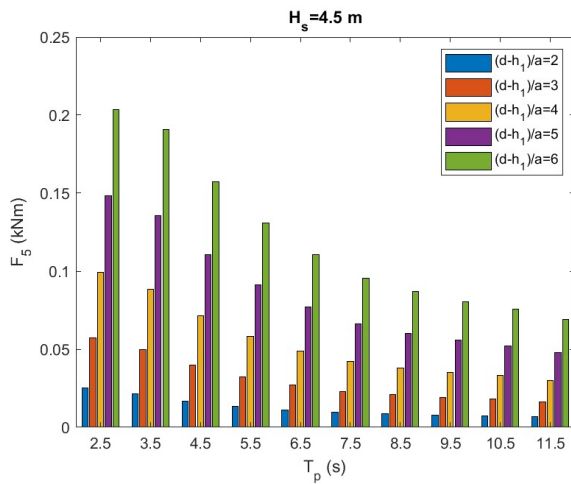
(b)



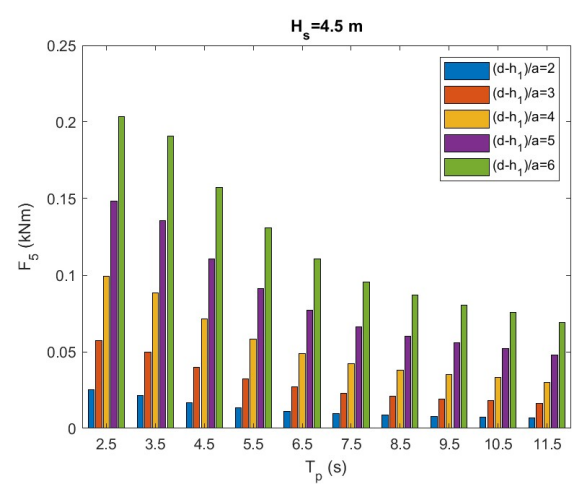
(c)



(d)



(e)



(f)

Figure 3.28: Values of  $F_5$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of draught

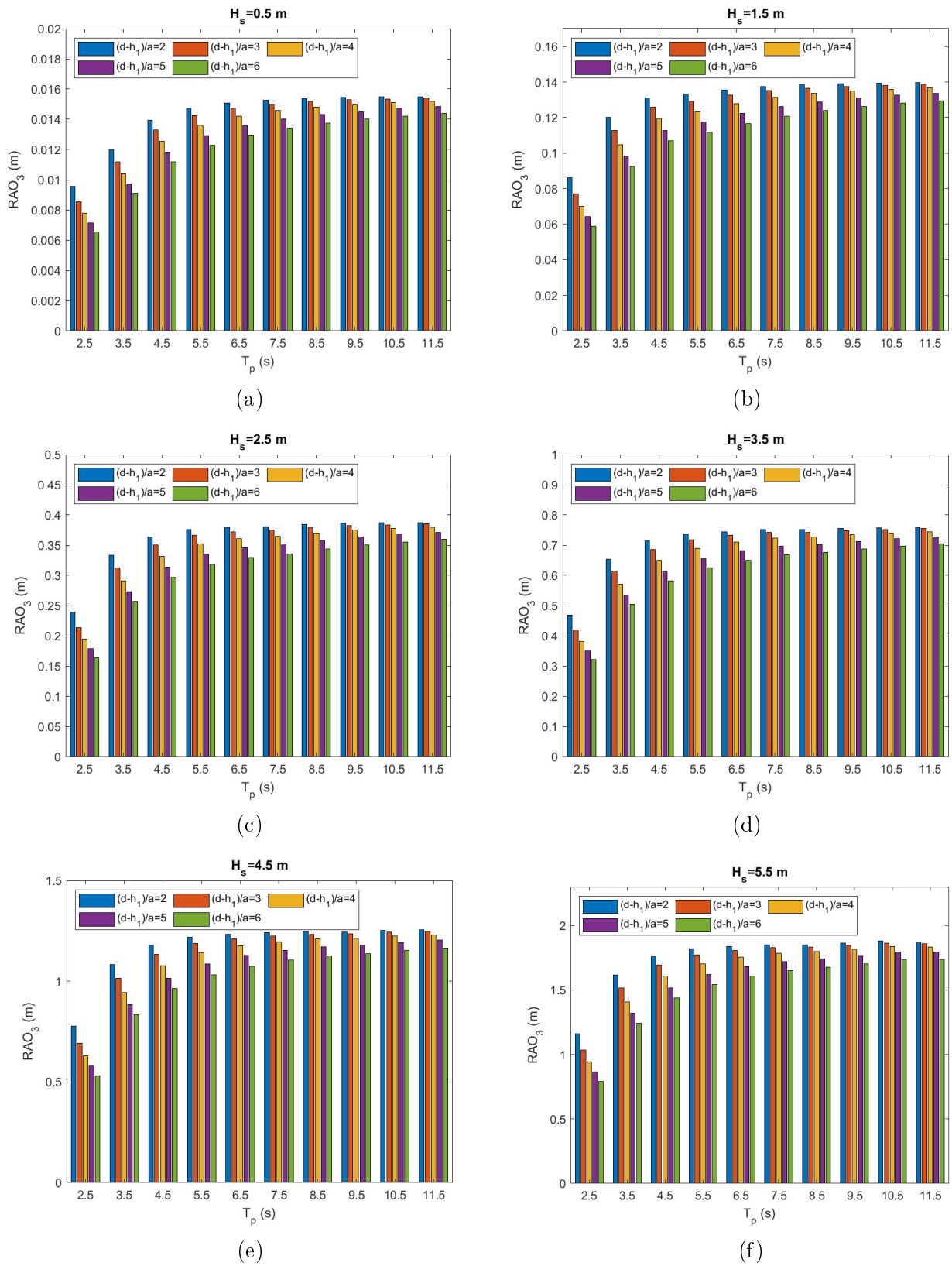


Figure 3.29: Values of  $RAO_3$  against peak wave period  $T_p$  and significant wave height  $H_s$  for various values of draught

## 4 Conclusions and Future Work

In the present a study a free floating, permeable cylinder body in the presence of regular waves was investigated. In order to investigate the hydrodynamics of the body, and within the realm of the linear theory, a three dimensional theoretical solution based on the eigenfunction method was developed for the determination of the velocity potential of the flow field around the cylindrical body. Furthermore, the linear resistance law was implemented to connect the the pressure drop across the porous boundary to normal velocity. The main conclusions drawn from this study are:

- The porous boundary contributes to the presence of reduced hydrodynamic forces and moment compared to those of the solid cylinder.
- The value of porous coefficient  $G$  plays a key role in the hydrodynamic forces and moment acting on the body as they decrease proportionally with its increase.
- The value of porous coefficient  $G$  is significant in the body's heave motion  $RAO$  as well as in its response in real waves.
- The overturning moment  $F_5$  increases proportionally with the body radius for all cases except  $r = 2a$ , this occurs due to the low damping values  $B_{55}$  for the case of  $r = 2a$  leading to higher values of exciting moment.

This research can be further expanded by taking into account the wave interaction between arrays of floating porous bodies. Additionally, it can be further developed in order to study floating porous bodies with flexible sidewalls, in order to simulate fish cages.

## References

## References

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